

THERMO
ANALYTICS



Reduce Vehicle Development Costs with Real-world Simulations

Jon Juskiewicz, Thermal/CFD Engineer



Agenda

1. Modern Solutions to Age Old Problems – The need for transient simulation
 1. Growth is Good
 2. How Do Engineers Create Growth Opportunities?
 3. Vehicle Thermal Test Cycles
 4. Modern Design Targets
2. The Million Dollar Question and How to Answer It
 1. The \$1M Question
 2. Modeling Options
3. A Novel Approach to Drive Cycle Simulation – Theory, History and a Commercial Solution
 1. Theory
 2. Background
 3. A Commercial Solution
4. Case Study – Strategies for Simulating Test Cycles in 3D
5. Conclusions

Modern Solutions to Age Old Problems – The need for transient simulation

Growth is Good

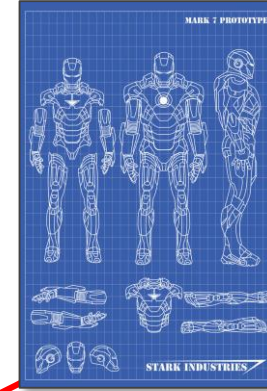
Happy business people



Higher pay



More interesting projects

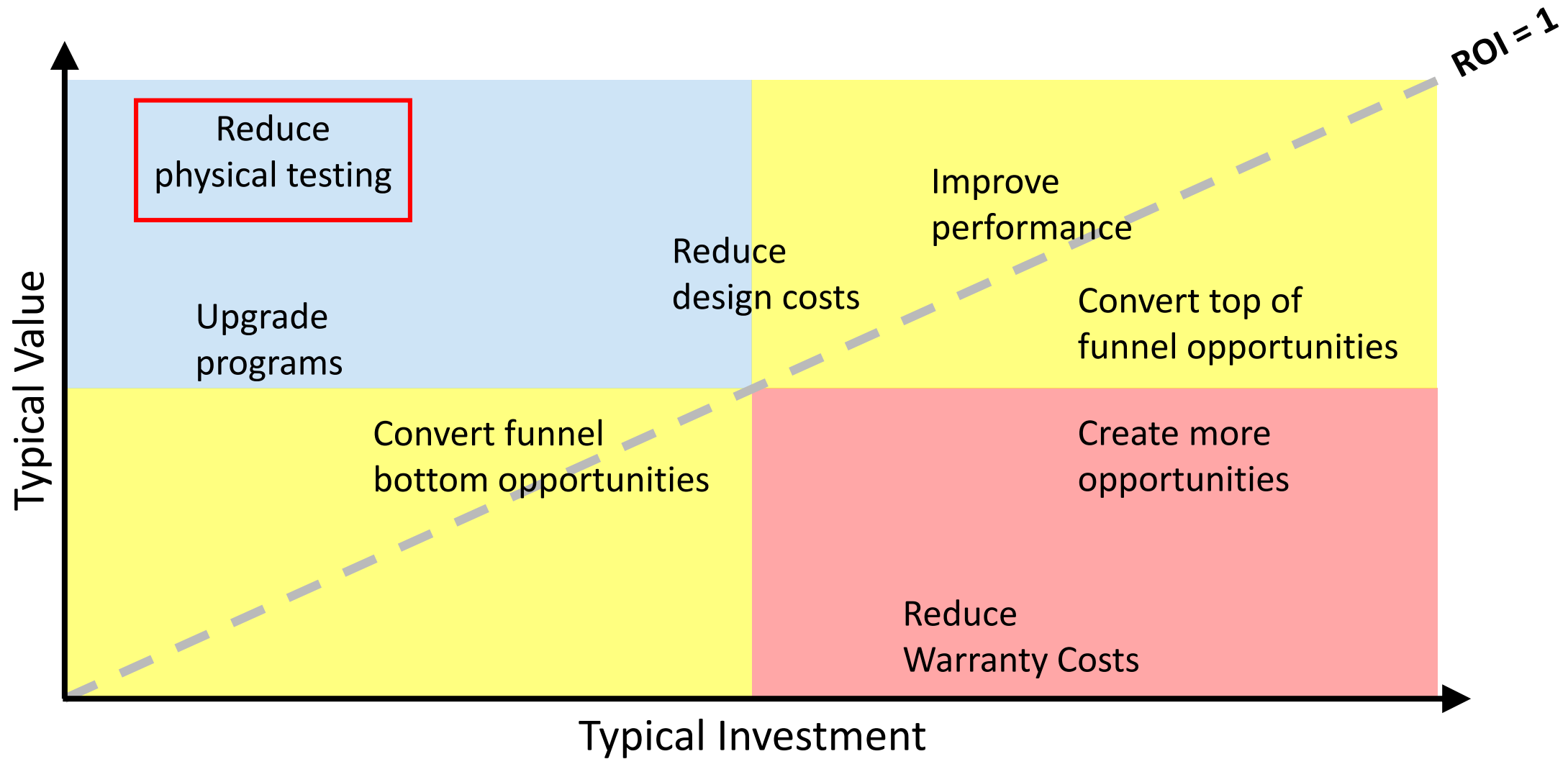


Happy Engineers

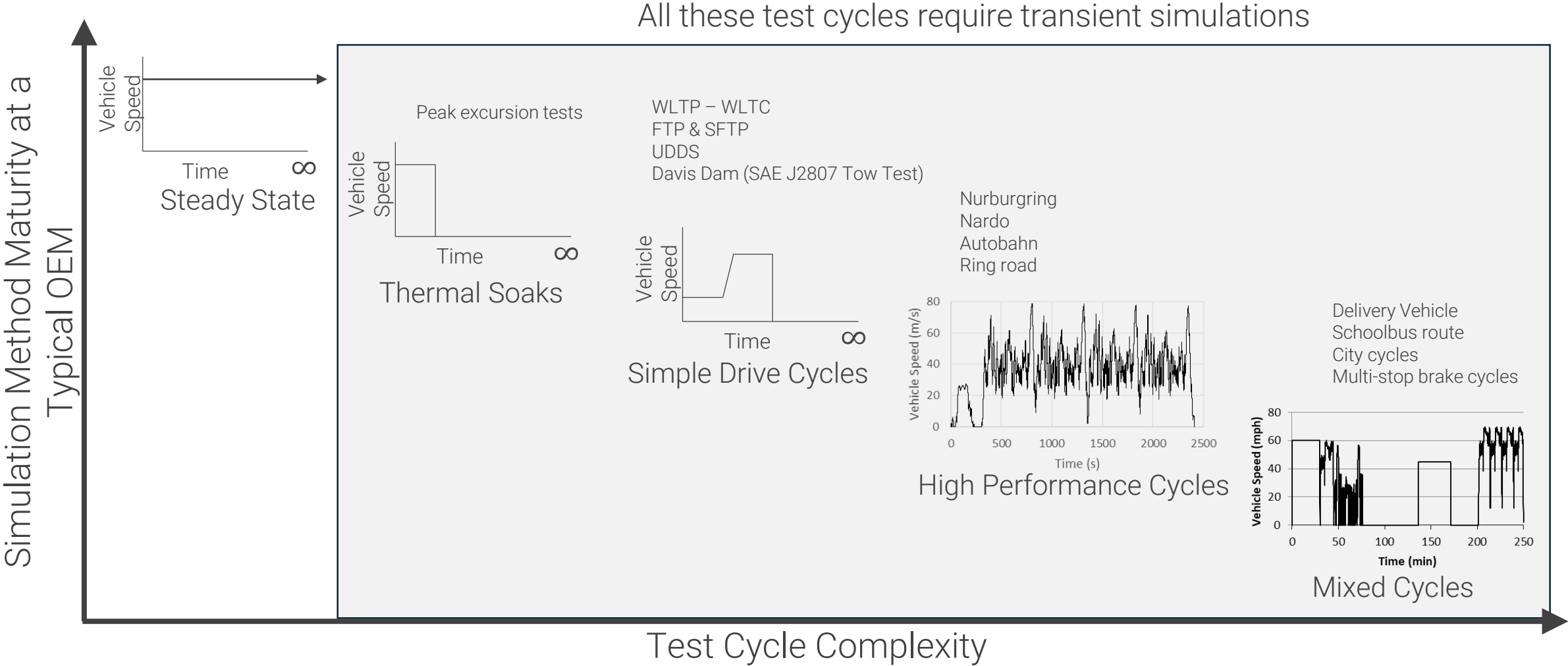


Increased investments

How can Engineers Create Growth?



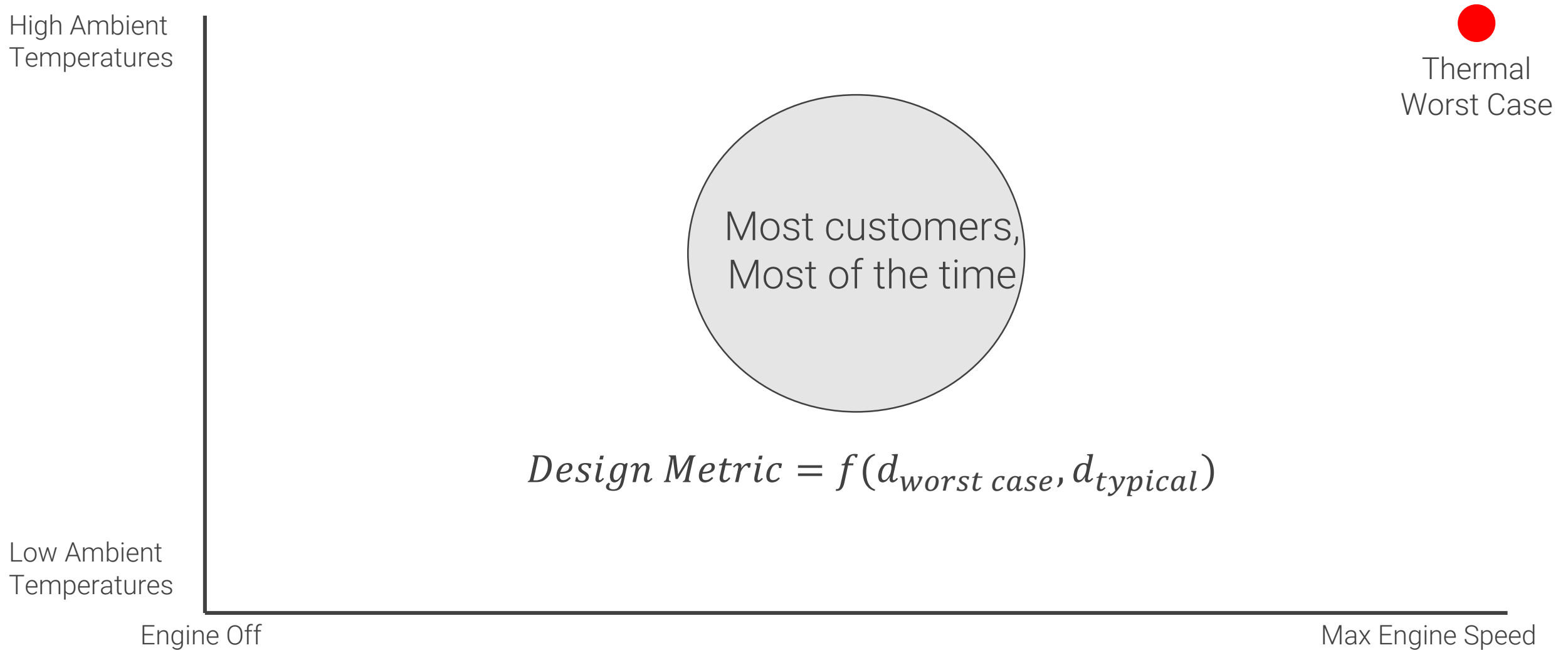
Vehicle Thermal Test Cycles



Engineering Approximations
of End Customer Behavior

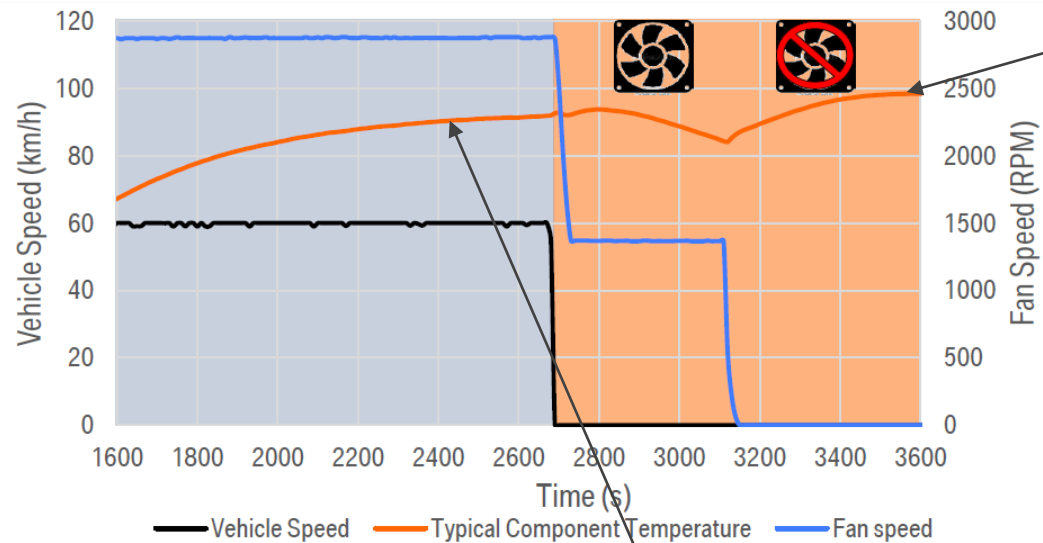
What End Customers
Actually Do

Modern Design Targets



Modern Design Targets

High Temp

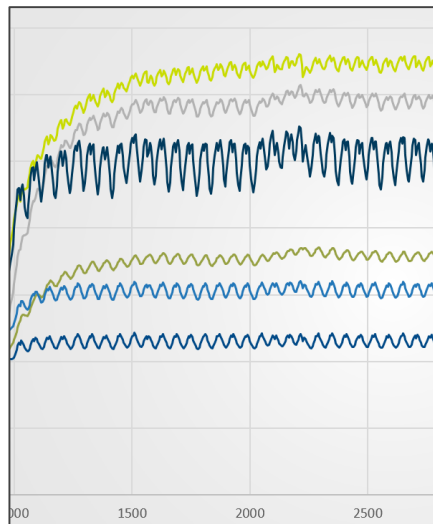


Thermal Worst Case (150 °C)

Does the fan induced cycling or thermal shock cause more part failures?

Thermal Worst Case

Low Ambient Temperature



Fan induced temperature cycling (110-130 °C)

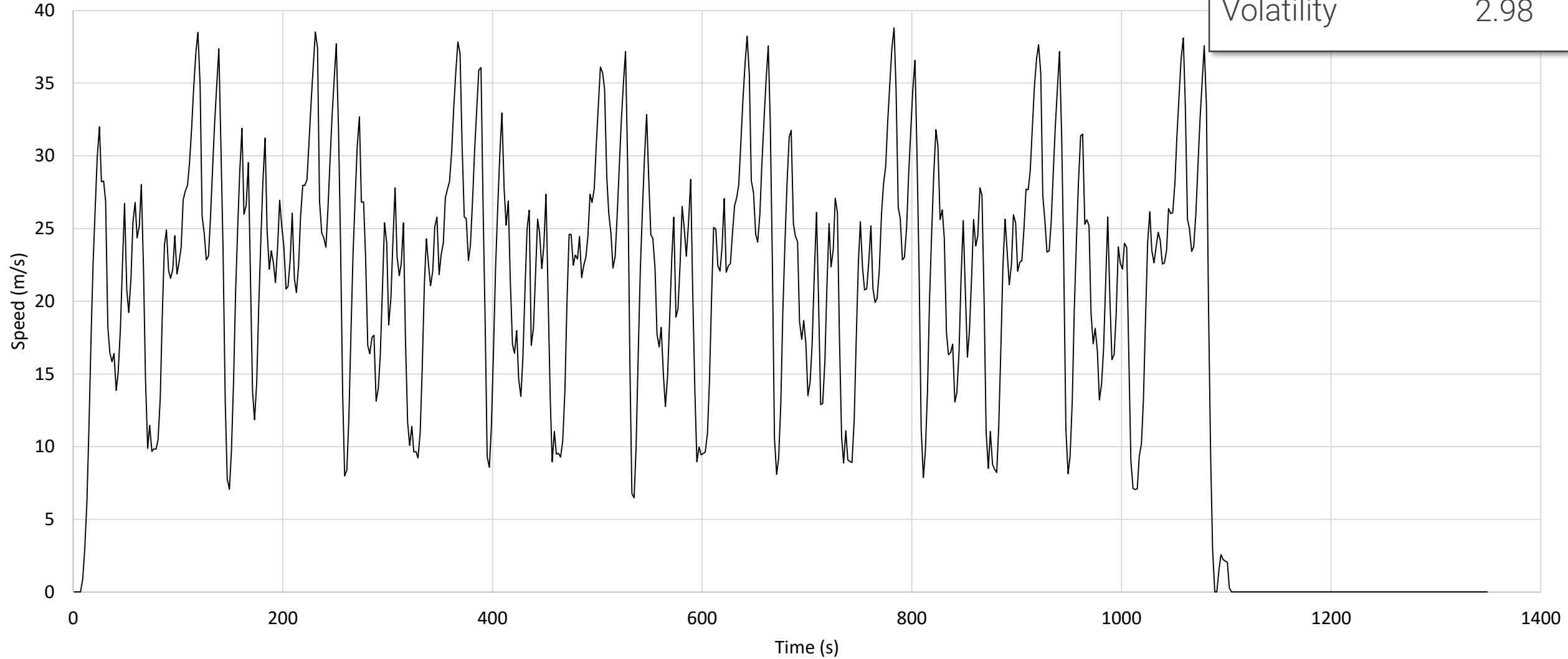
worst case, $d_{typical}$)

Max Engine Speed

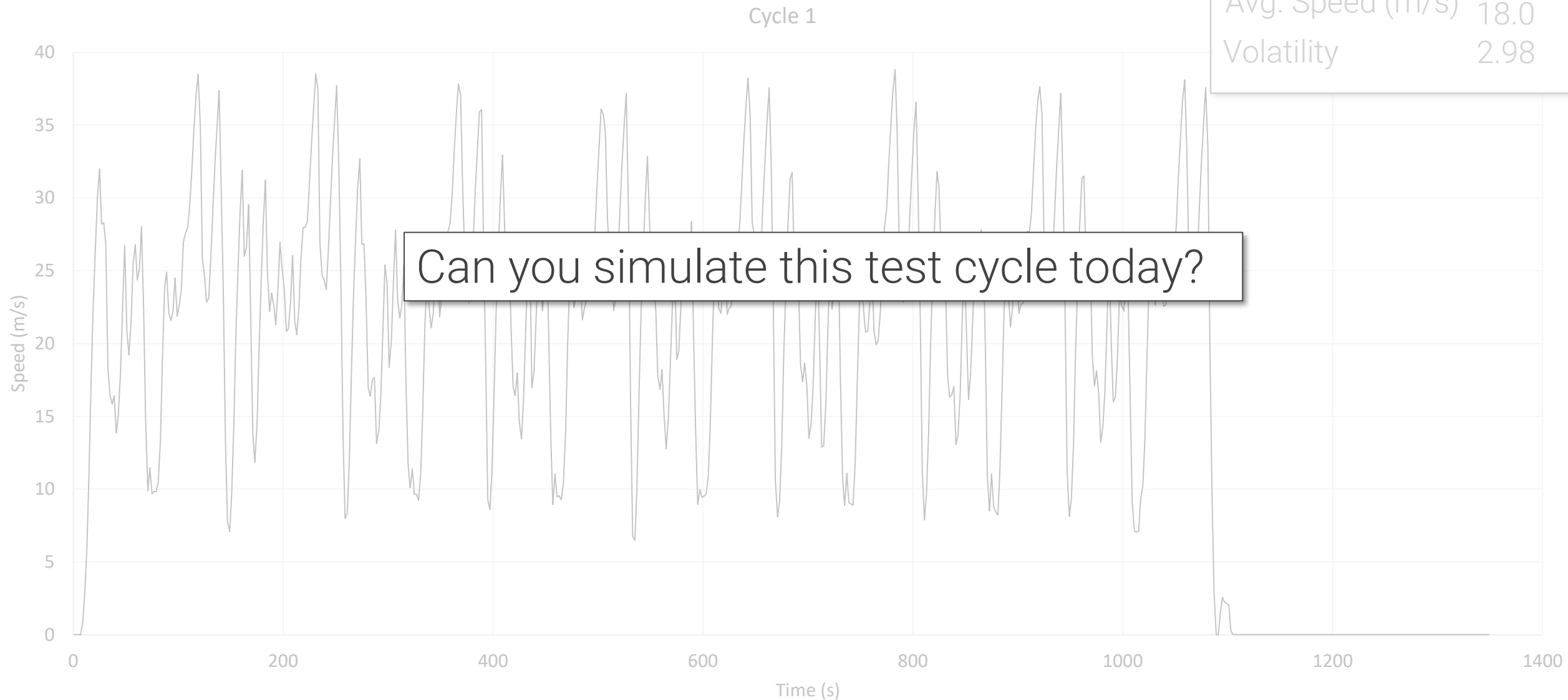
The Million Dollar Question and How to Answer It

The \$1M Question

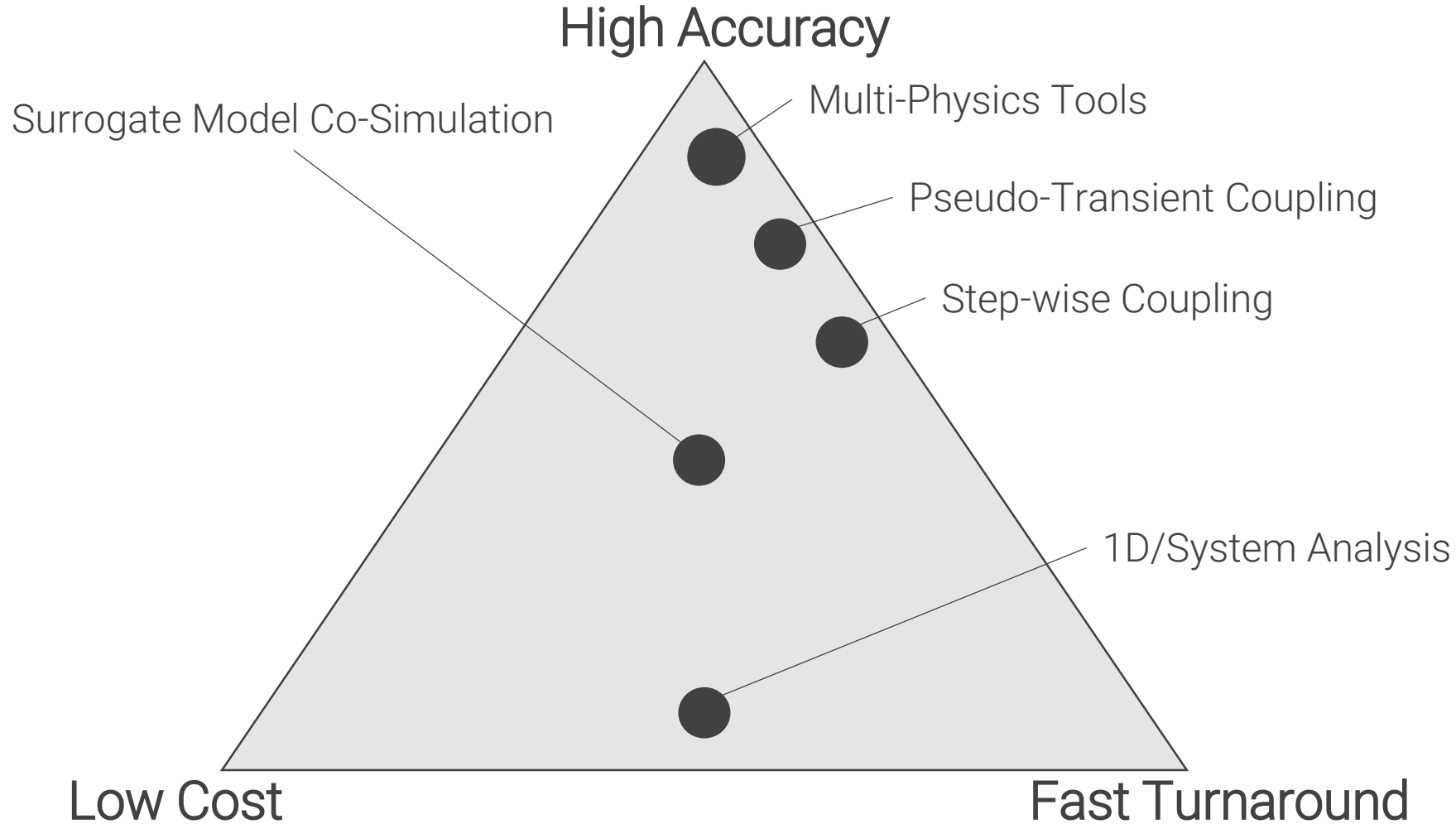
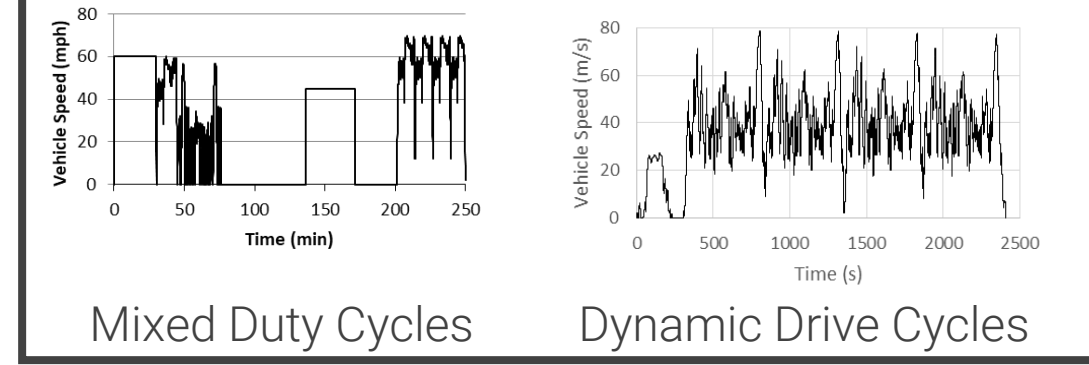
Cycle 1



The \$1M Question

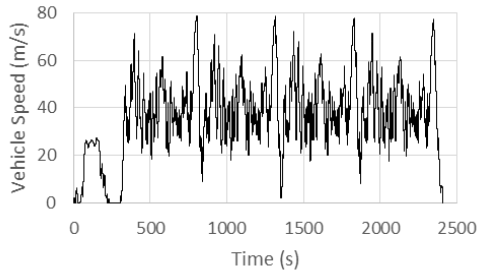


Modeling Options

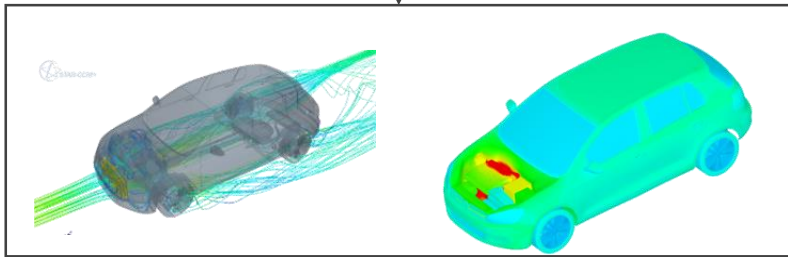


A Novel Approach to Drive Cycle Simulation – Theory, History and a Commercial Solution

Theory



Select a set of representative vehicle parameters



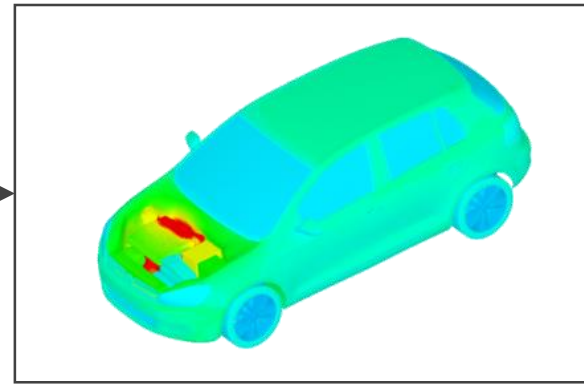
Calculate the steady state CHT solution at each selected speed

Extract HTC & T_f

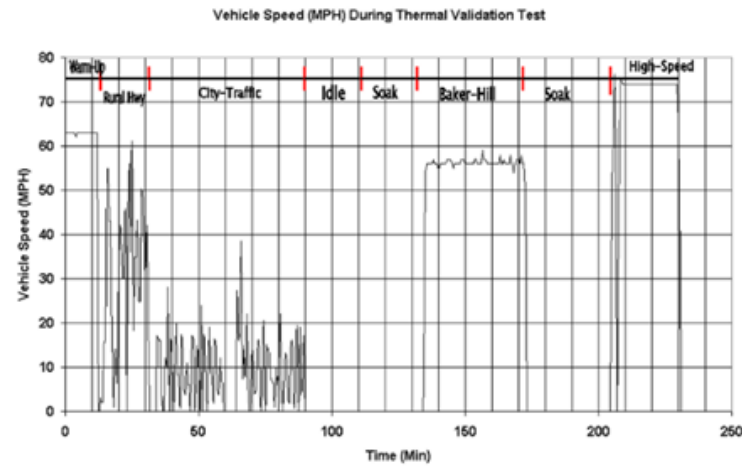
Surrogate Model

Make the samples continuous over the range of speeds with a surrogate model

Solve transient thermal simulation



Background

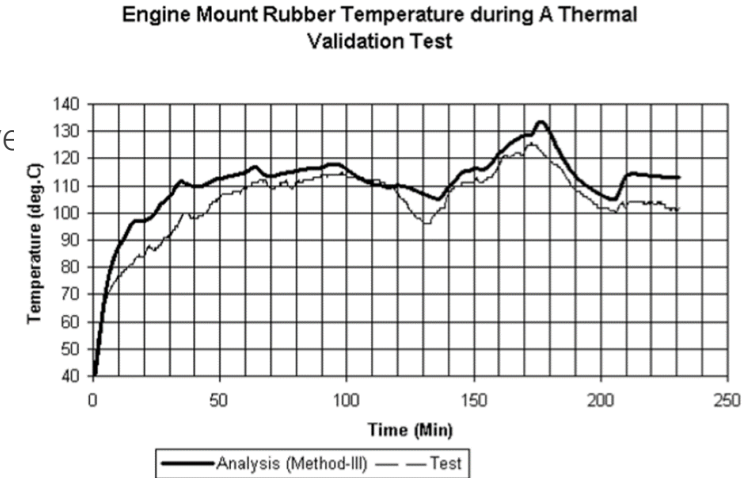


1. Vehicle Speed
2. Fan Speed
3. Radiator Exit Temperature
4. Ambient Air Temperature
5. Exhaust Gas Flow Rate
6. Exhaust Gas Temperature

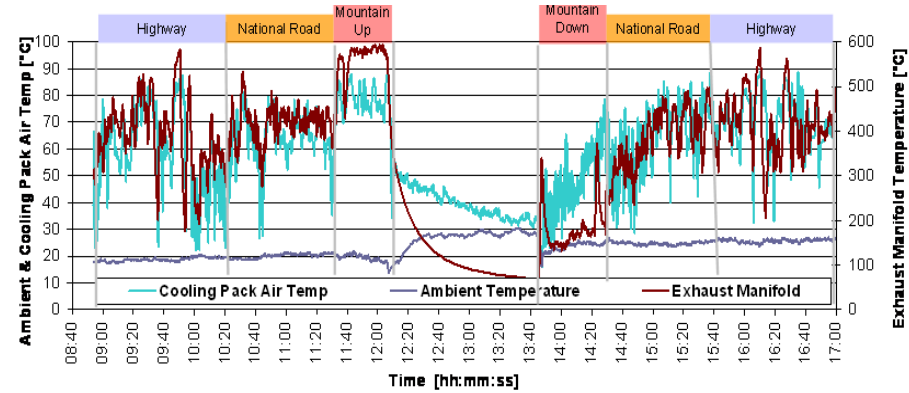
OLHC
Kriging

Part Average HTC Curve

Part Average Tf Curve



Background



Vehicle Speed

Fan Mass
Flow Rate

Fan Air
Temperature

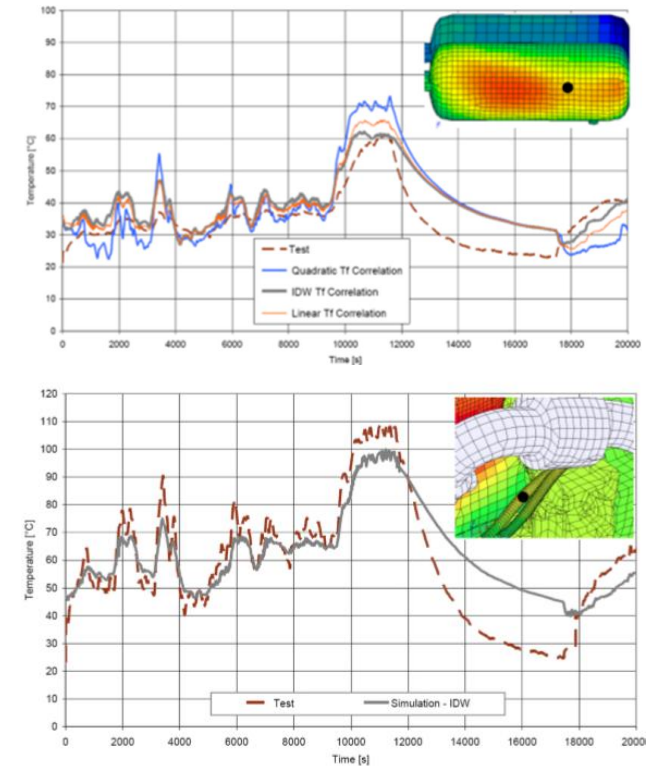
Exhaust Gas
Temperature

Manual
Selection

IDW

Local HTC
Surrogate Model

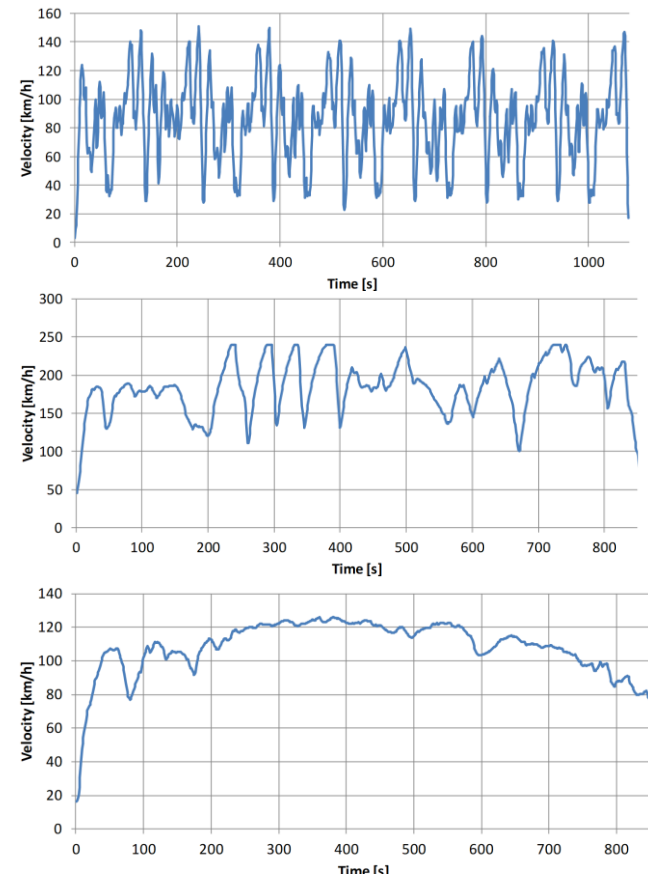
Local Tf Surrogate
Model



2007
Kaushik
GM Research

2011
Pryor
ThermoAnalytics

Background



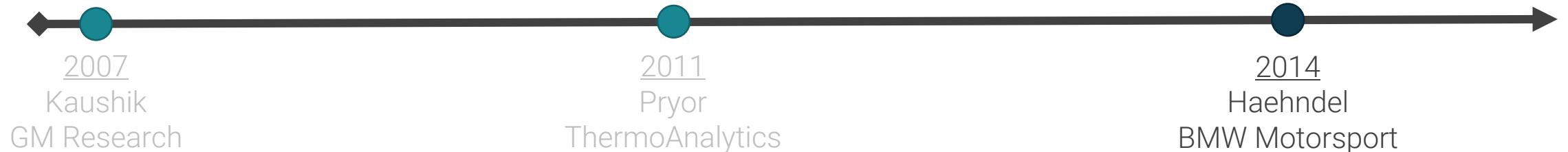
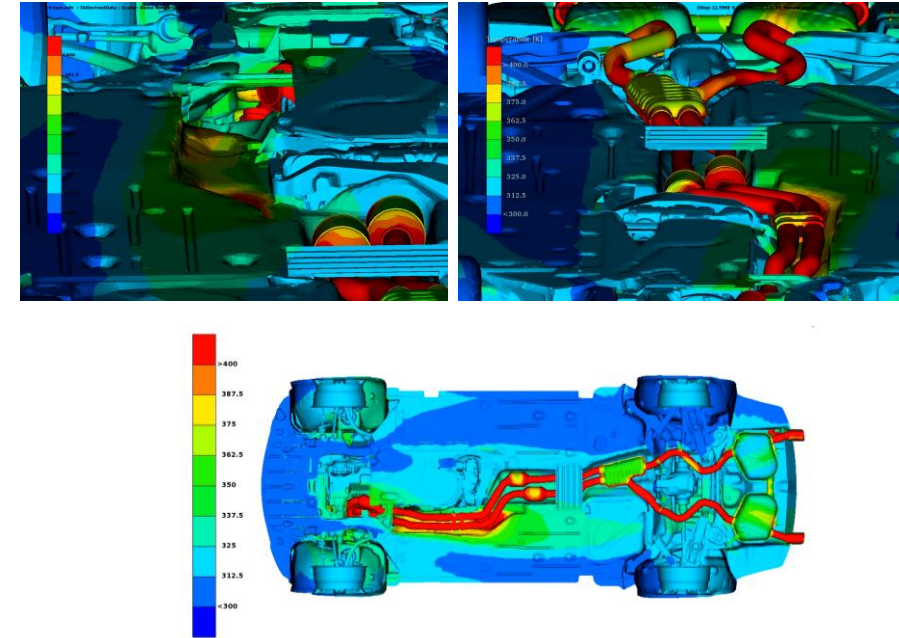
Vehicle Speed

Undisclosed

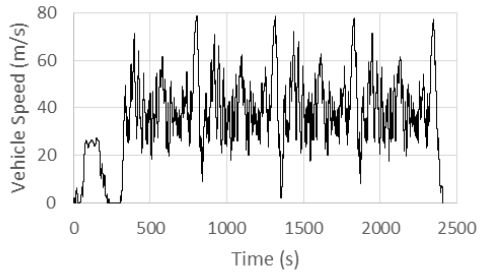
Linear

Local HTC
Surrogate Model

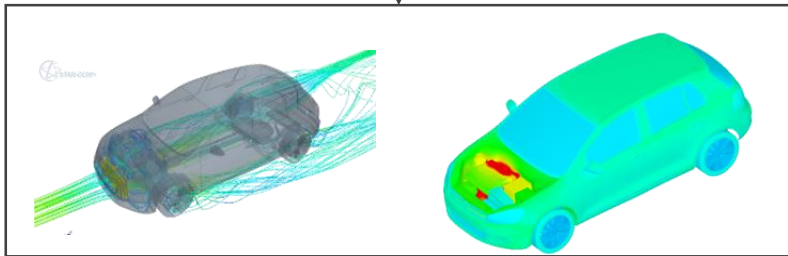
Local Tf Surrogate
Model



A Commercial Solution



↓
Select a set of representative
vehicle parameters



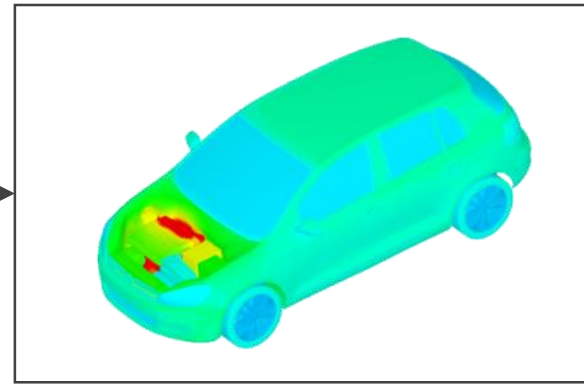
Calculate the steady state CHT
solution at each selected speed

Extract HTC
& T_f

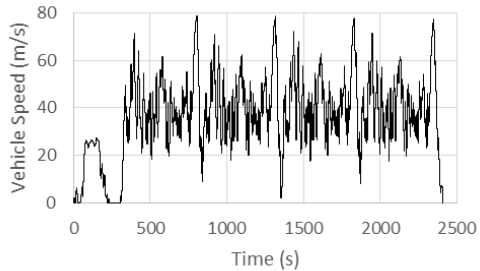
Surrogate Model

Make the samples continuous
over the range of speeds with a
surrogate model

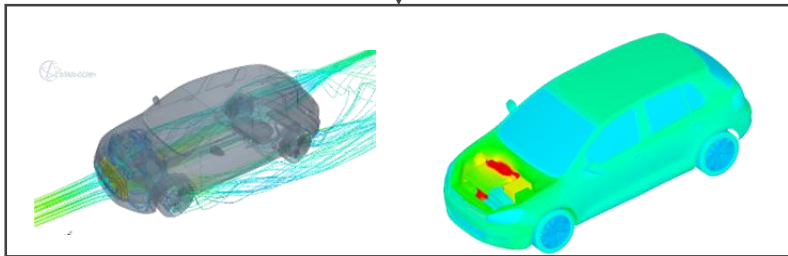
Solve transient thermal
simulation



A Commercial Solution



↓
Select a set of representative
vehicle parameters



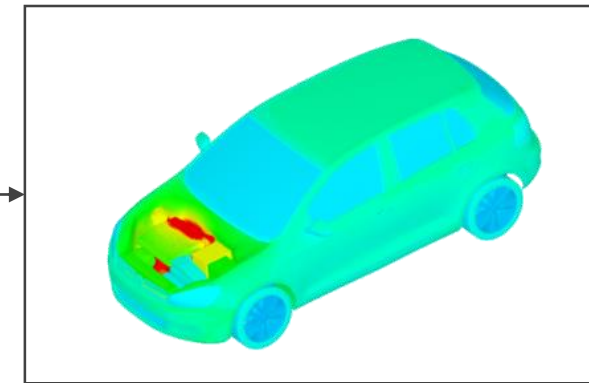
Calculate the steady state CHT
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Extract HTC
& T_f

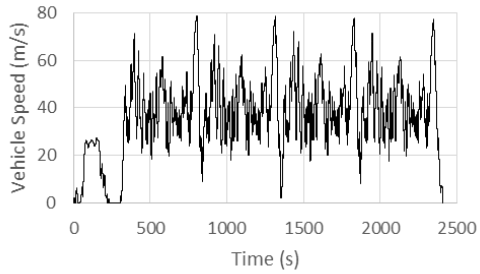


Make the samples continuous
over the range of speeds with a
surrogate model

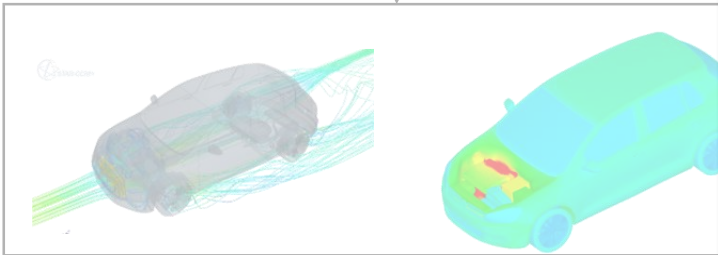
Solve transient thermal
simulation



A Commercial Solution



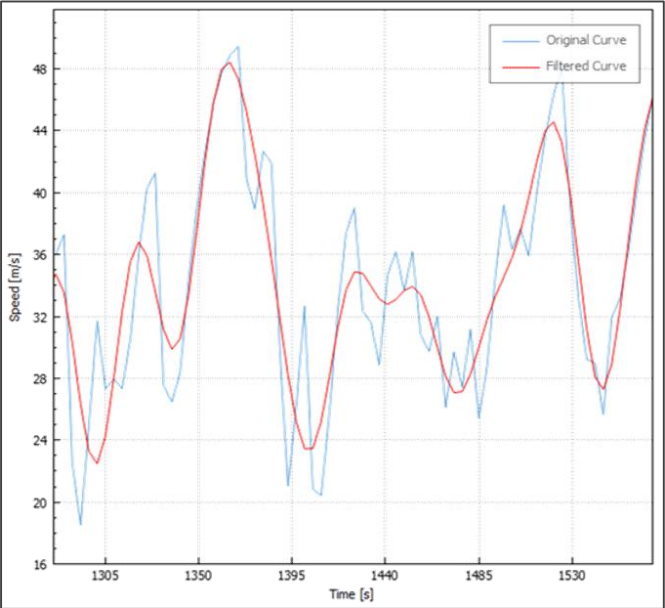
Select a set of representative vehicle parameters



Calculate the steady state CHT solution at each selected speed

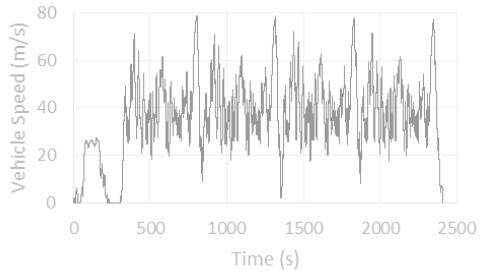
FILE HOME INSERT PAGE LAYOUT FORMULAS DATA REVIEW VIEW POWER QUERY					
Clipboard Font Alignment Number Styles					
A1 Time (s)					
A	B	J	L	N	
1	Time (s)	Speed (m/s)	Ex. Mass Flow Rate (kg/s)	Ex. Temperature (K)	Engine Speed (RPM)
2	1.0000209	0.008913901	0.025552945	375.67374	793.4736
3	2.0000207	0.009624188	0.021182489	369.7271	867.8779
4	3.0000207	0.007663409	0.019731915	367.93878	822.34686
5	4.000021	0.009535631	0.02180889	371.62613	861.0781
6	5.000021	0.007328299	0.020332338	371.9891	718.8925
7	6.000021	0.00938529	0.019084198	366.48618	872.0943
8	7.000021	0.009032463	0.025420344	375.3413	803.90466
9	8.000021	0.007825852	0.019962268	366.79935	837.5564
10	9.000021	0.8882079	0.019453956	366.89545	869.021
11	10.000021	2.0258005	0.028914375	444.31354	1209.368
12	11.000021	3.0039587	0.02818598	414.7149	1331.8081
13	12.000021	4.003674	0.041972242	486.5666	1777.5978
14	13.000021	6.437684	0.070185035	669.8015	2081.262
15	14.000021	8.766538	0.10389075	677.5965	2404.4924
16	15.000021	11.387993	0.15551451	796.70215	3050.987
17	16.000021	14.627627	0.1835707	821.17523	3136.3518
18	17.000021	17.753935	0.1766344	826.0358	3203.11
19	18.000021	20.33142	0.20743442	882.04333	3507.373
20	19.000021	22.740816	0.1759784	823.1057	3207.8508
21	20.000021	24.80209	0.1903357	871.7763	3332.6423

Test Cycle Data Specified Directly in Excel

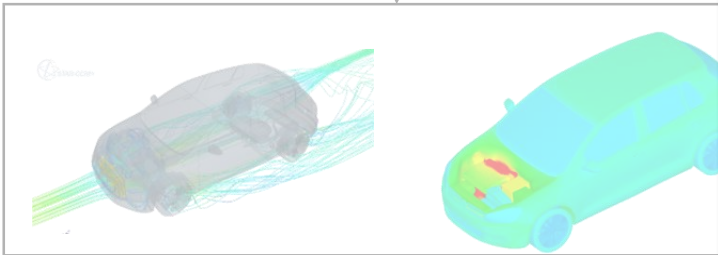


Process Data from Excel Directly in CoTherm

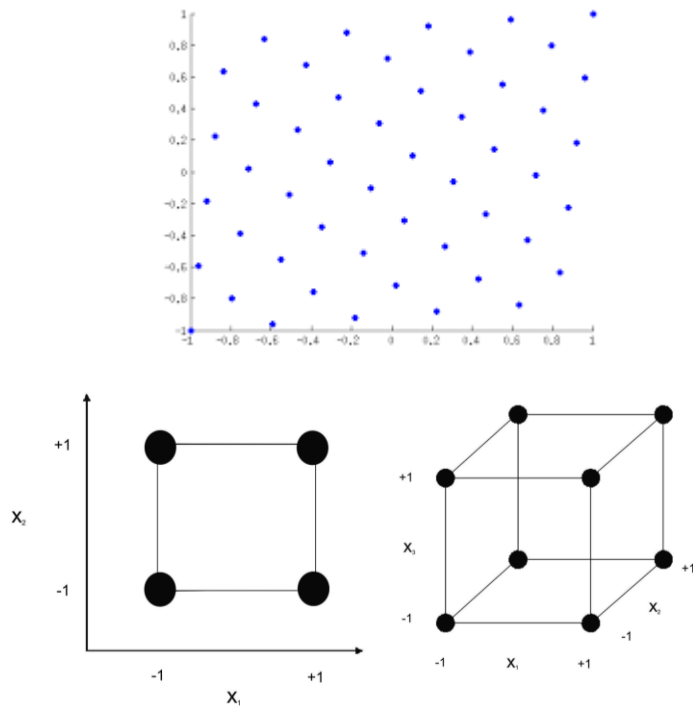
A Commercial Solution



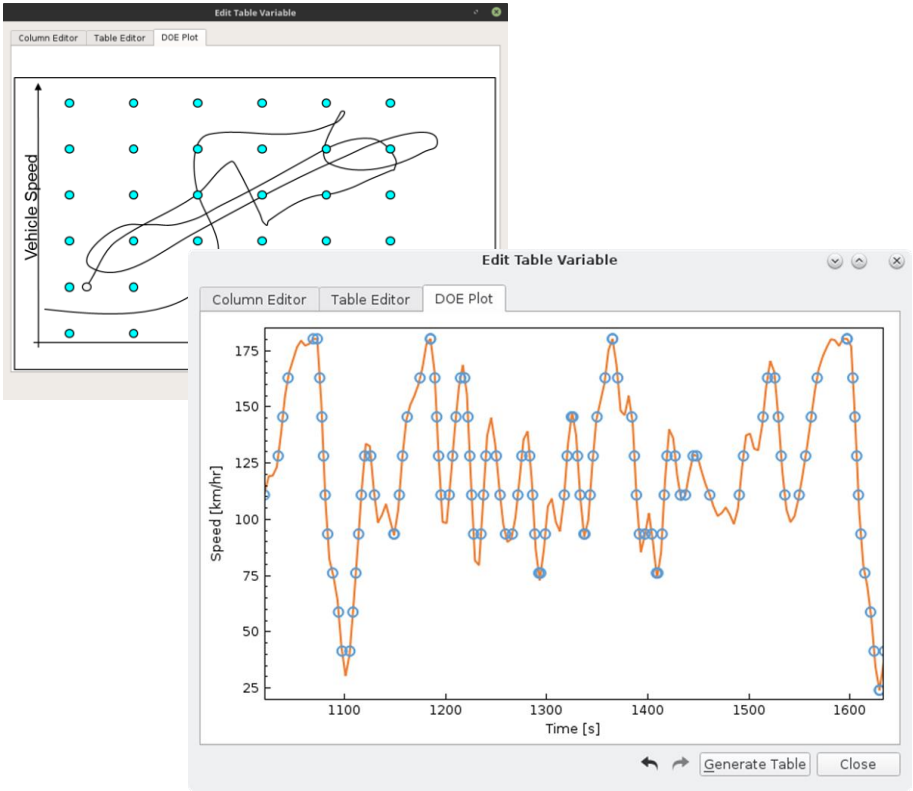
Select a set of representative vehicle parameters



Calculate the steady state CHT solution at each selected speed

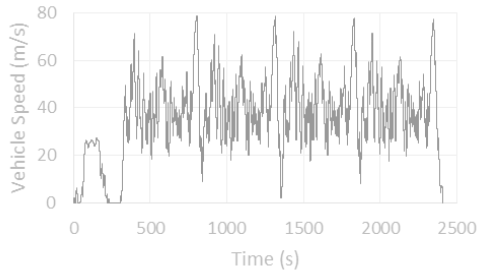


Select OLHC or Full Factorial Sampling Plans in Any Number of Dimensions

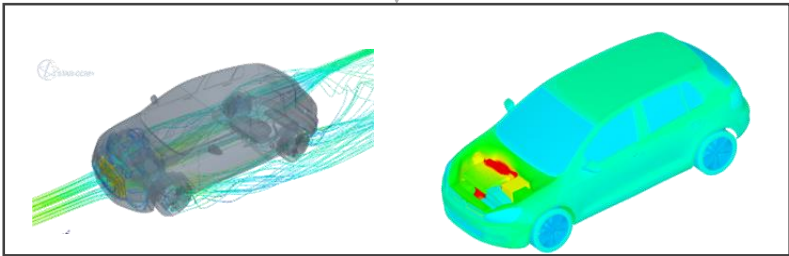


Visualize the Sample Points and Boundary Conditions

A Commercial Solution

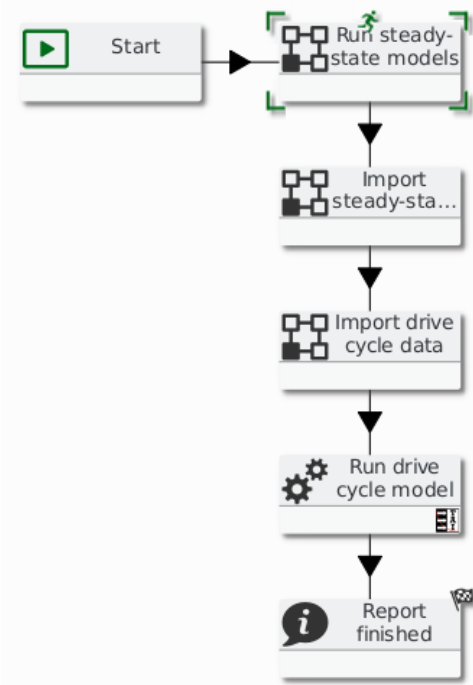


Select a set of representative vehicle parameters

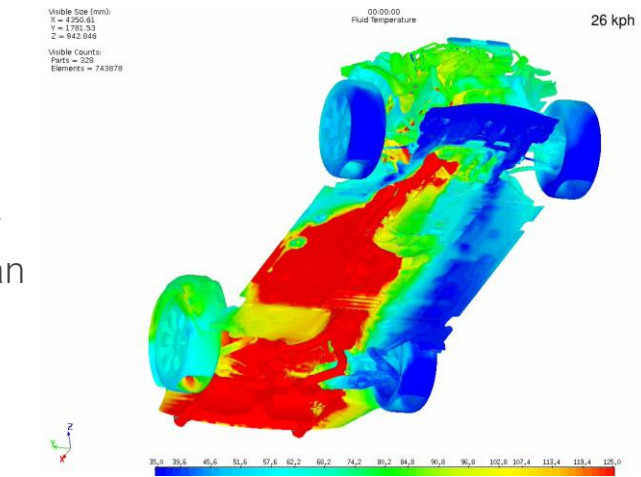
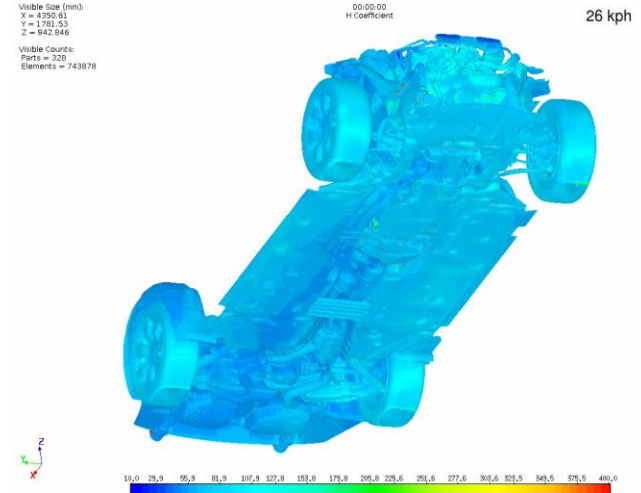


Calculate the steady state CHT solution at each selected speed

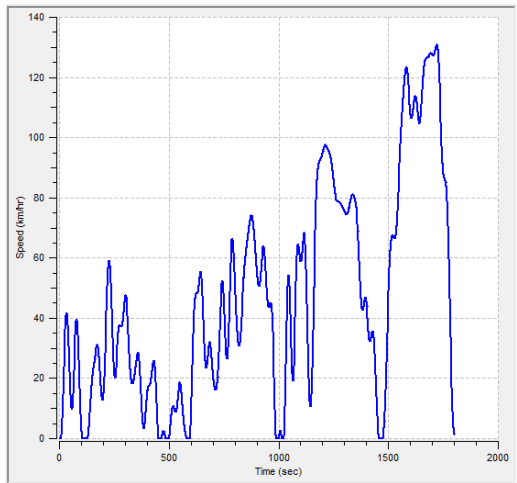
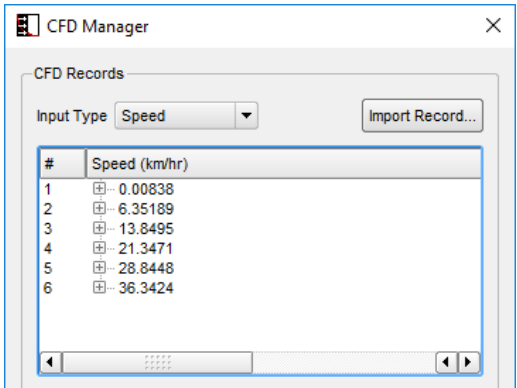
Ext



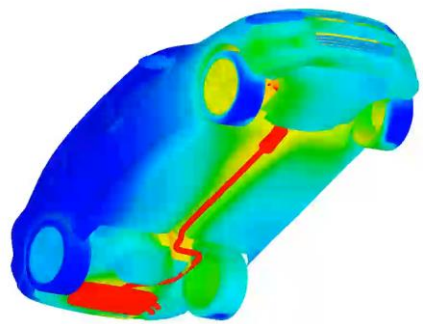
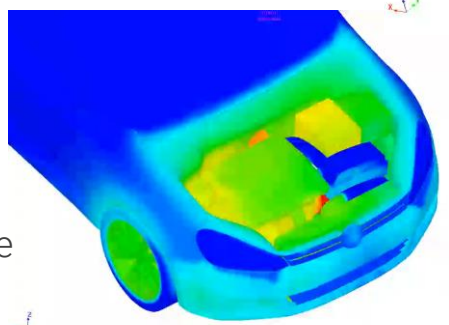
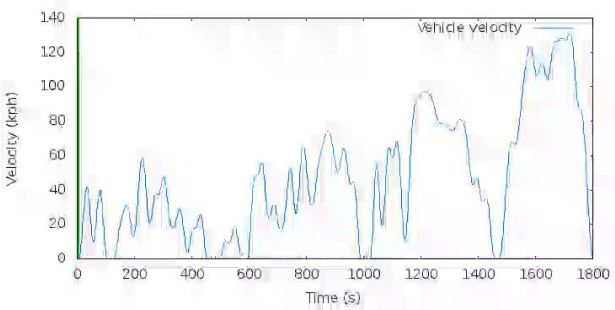
Sample Points are Automatically Calculated Based on Sampling Plan



A Commercial Solution

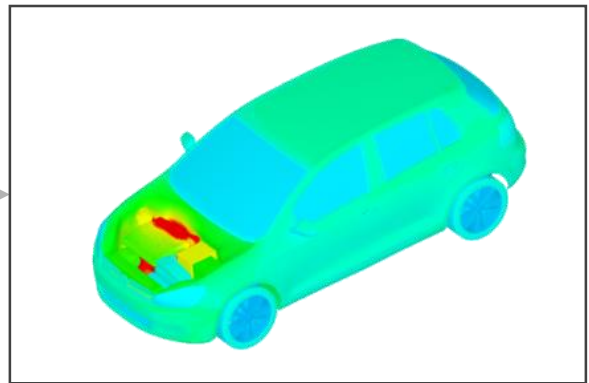


Automatically Create Surrogate Model and Import Boundary Conditions



Simulate Any Drive Cycle Bounded by the Surrogate Model Sample Points

Solve transient thermal simulation



Case Study –Strategies for Simulating Test Cycles in 3D

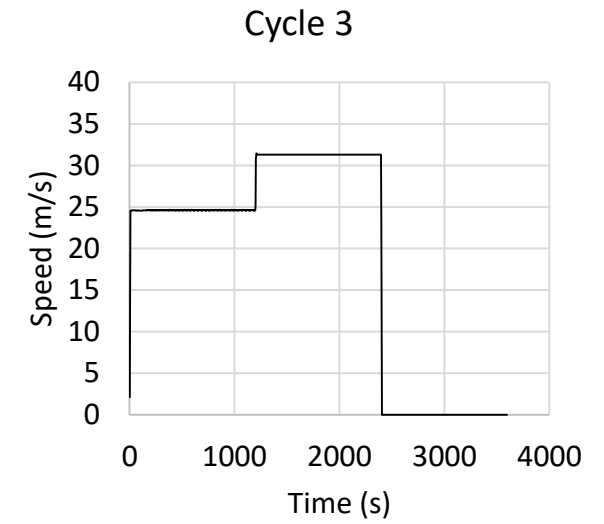
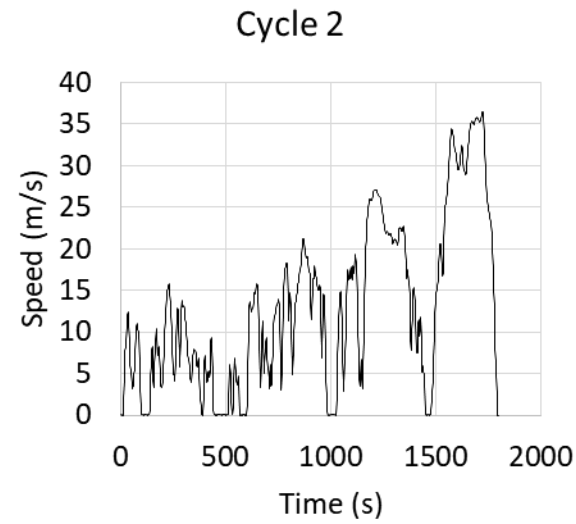
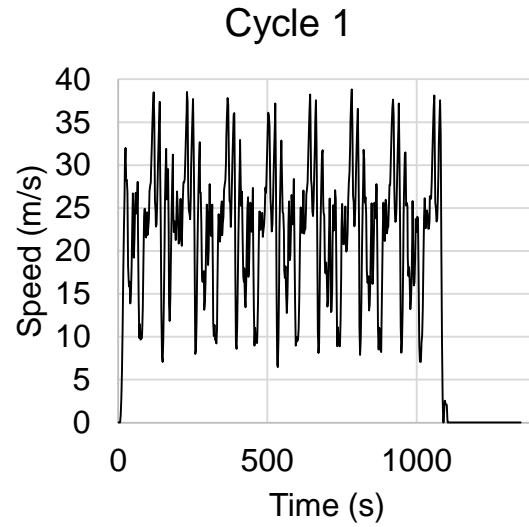
Case Study Introduction

3 Representative Speed profiles
Selected

Speed Profile

Speed Profile

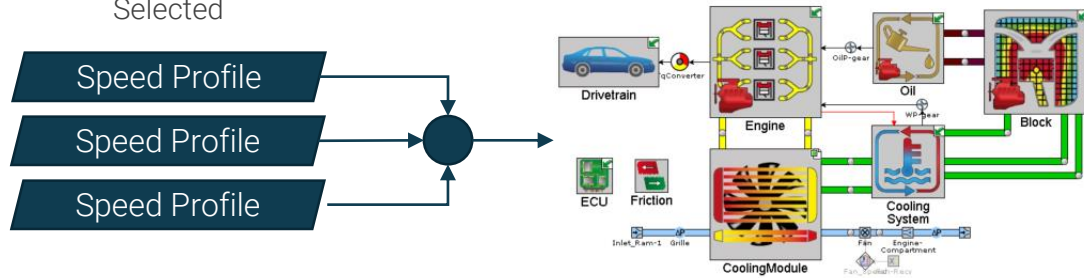
Speed Profile



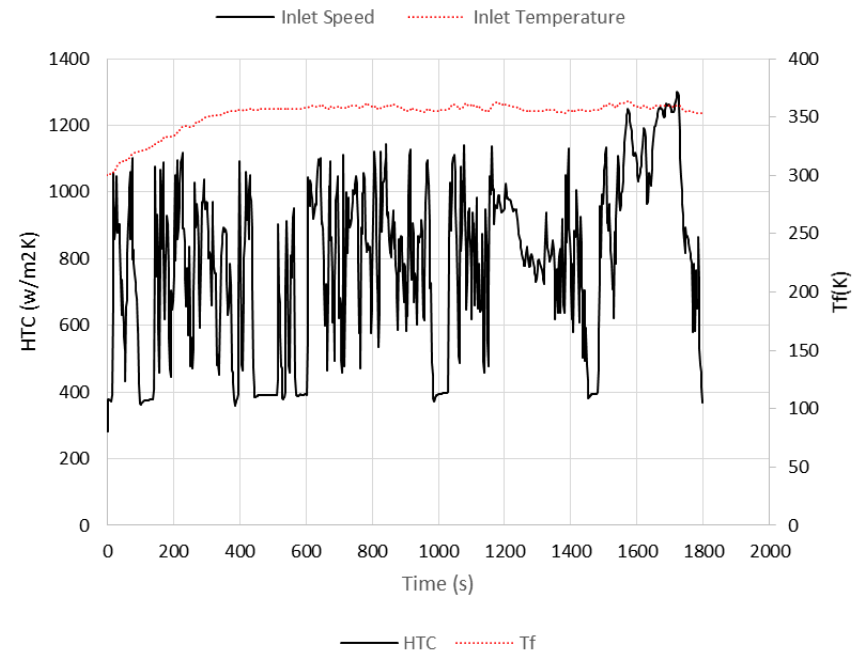
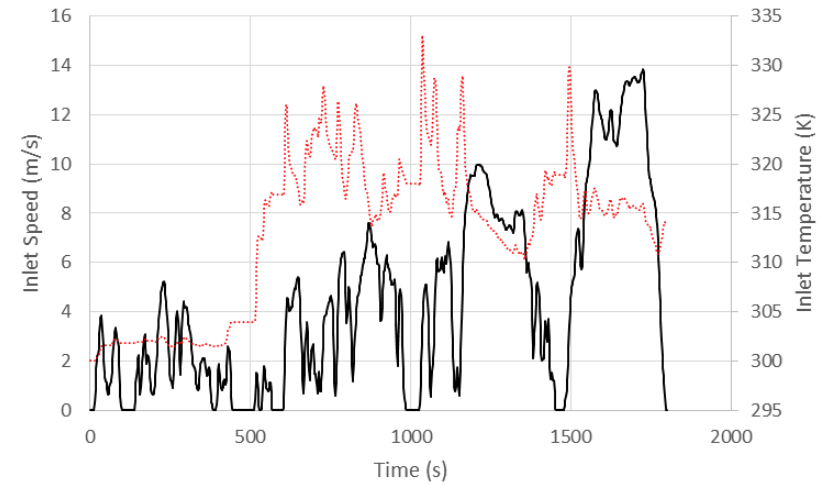
A Comparison of Strategies for Simulating Vehicle Heat Protection Test Cycles in 3D, SAE TMSS, 2018

Case Study Introduction

3 Representative Speed profiles
Selected



Representative 3D Model
Boundary Conditions Created
with GT-ISE System Model

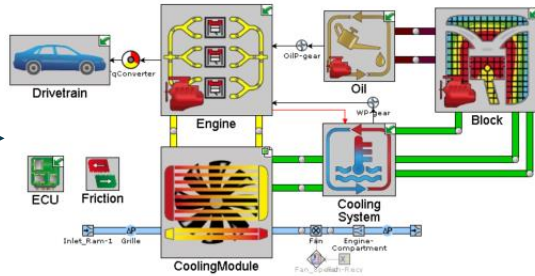


A Comparison of Strategies for Simulating Vehicle Heat Protection Test Cycles in 3D, SAE TMSS, 2018

Case Study Introduction

3 Representative Speed profiles
Selected

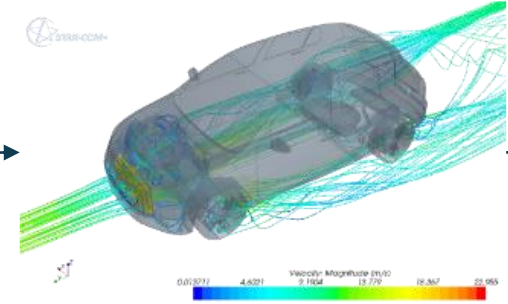
Speed Profile
Speed Profile
Speed Profile



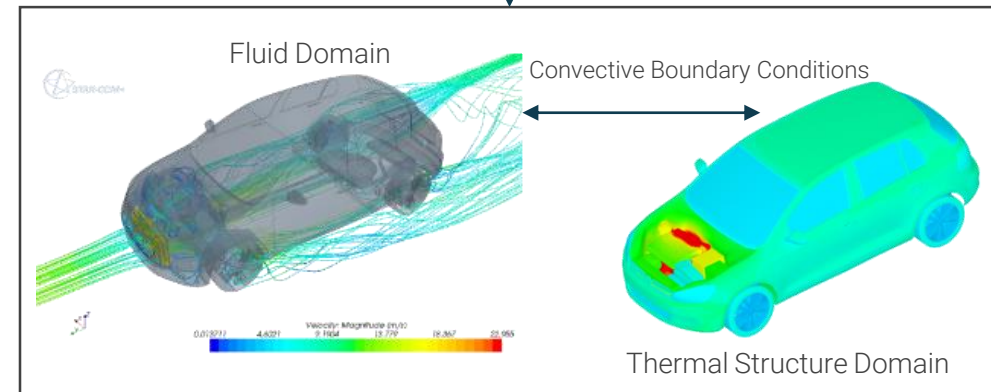
Representative 3D Model
Boundary Conditions Created
with GT-ISE System Model

Test Cycle
Boundary
Conditions

Establish a "True" Solution using a CHT simulation



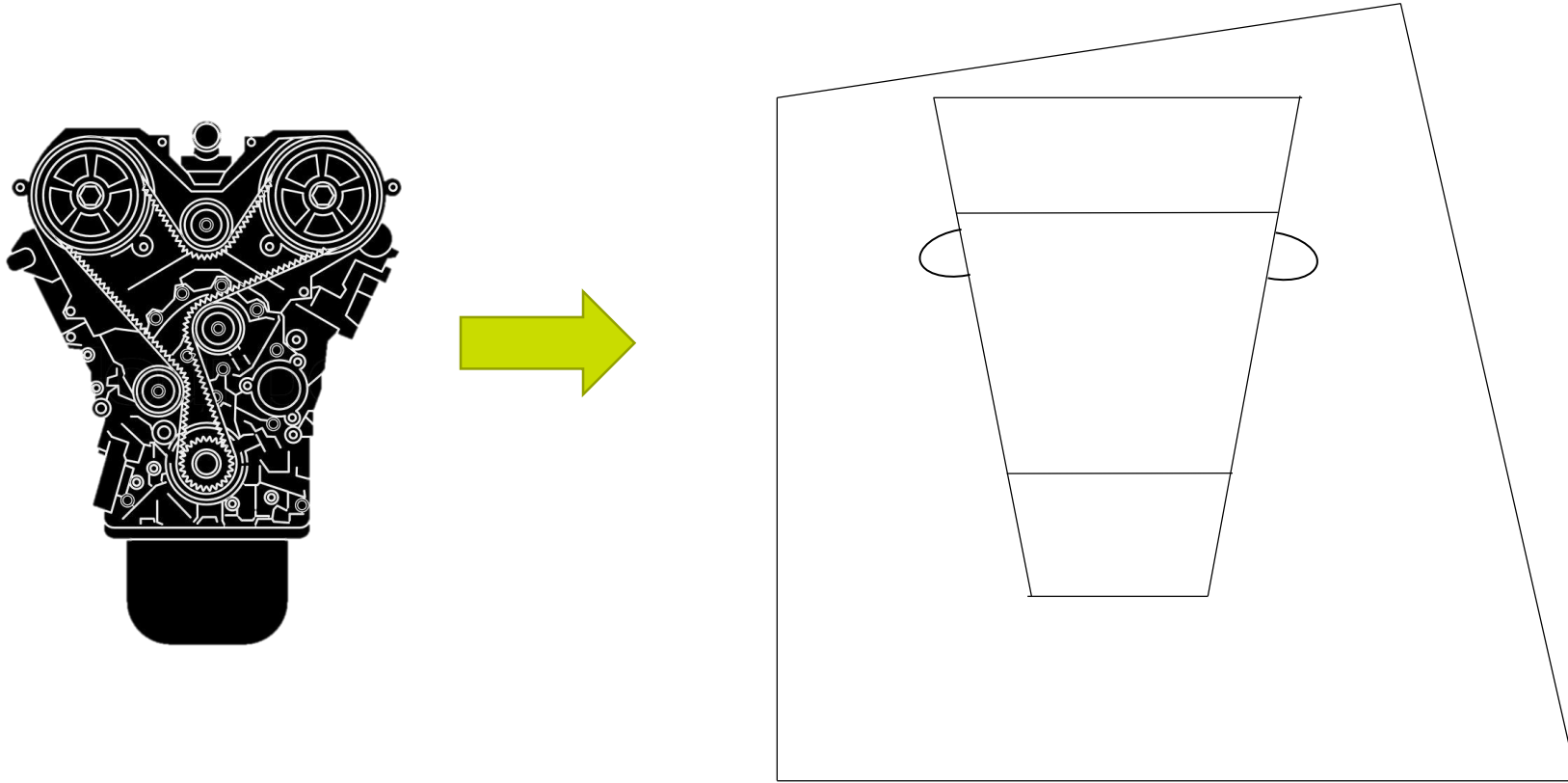
Compare CHT and Coupled Solutions



Simulate the Same Test Cycles With Segregated Domains Using Various
Coupling Strategies

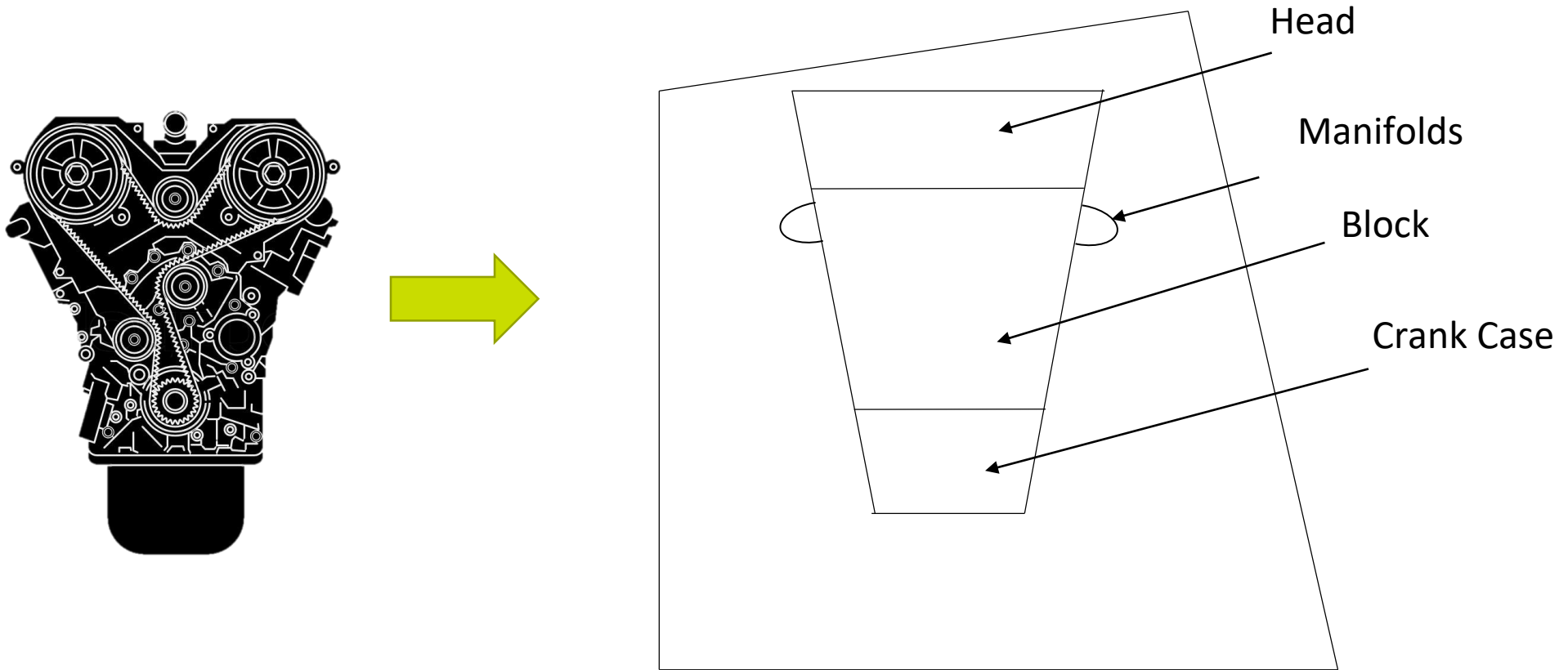
A Comparison of Strategies for Simulating Vehicle Heat Protection Test Cycles in 3D, SAE TMSS, 2018

Case Study Introduction

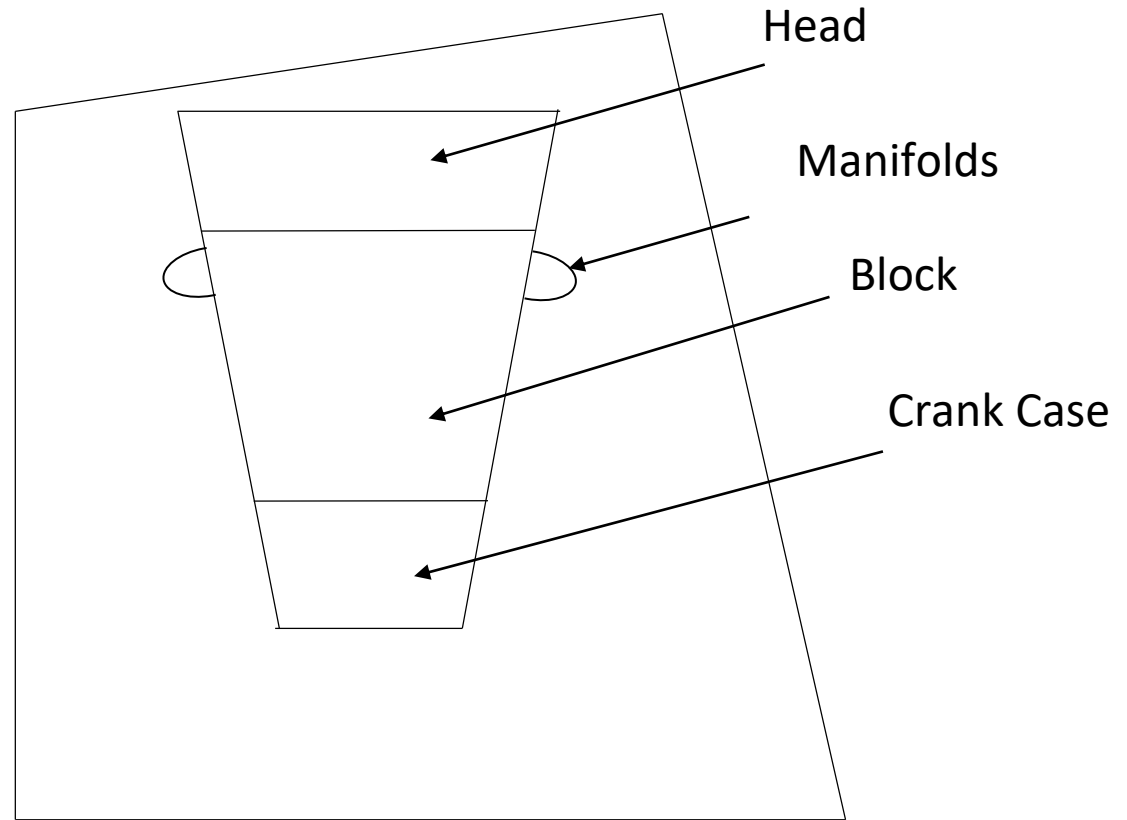
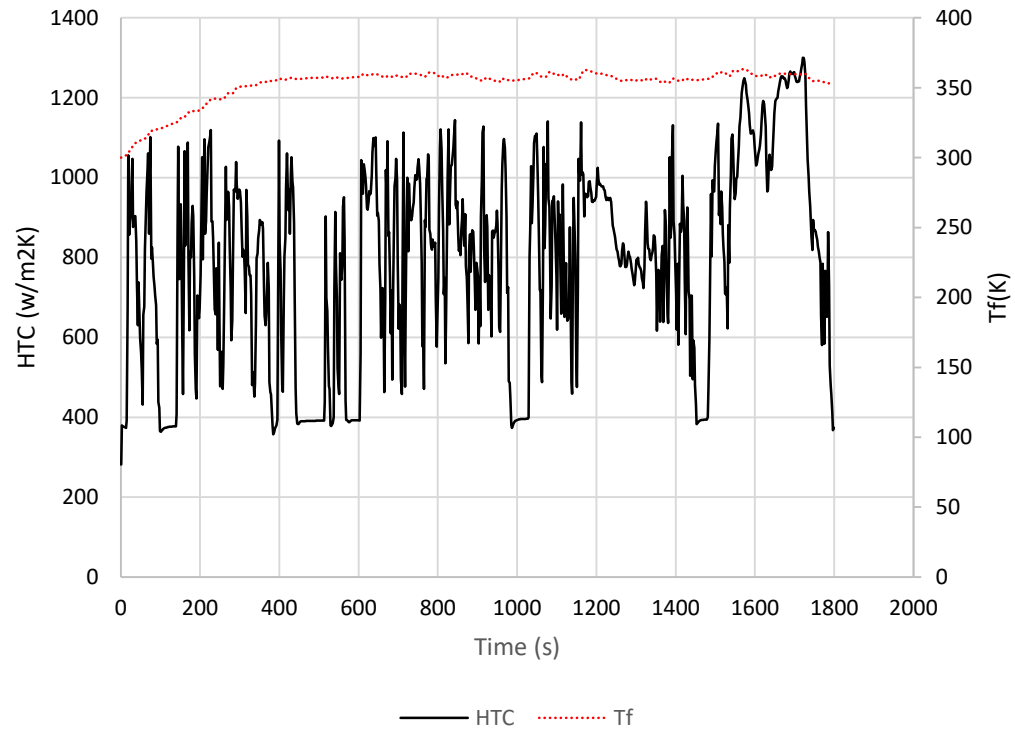


A Comparison of Strategies for Simulating Vehicle Heat Protection Test Cycles in 3D, SAE TMSS, 2018

Case Study Introduction

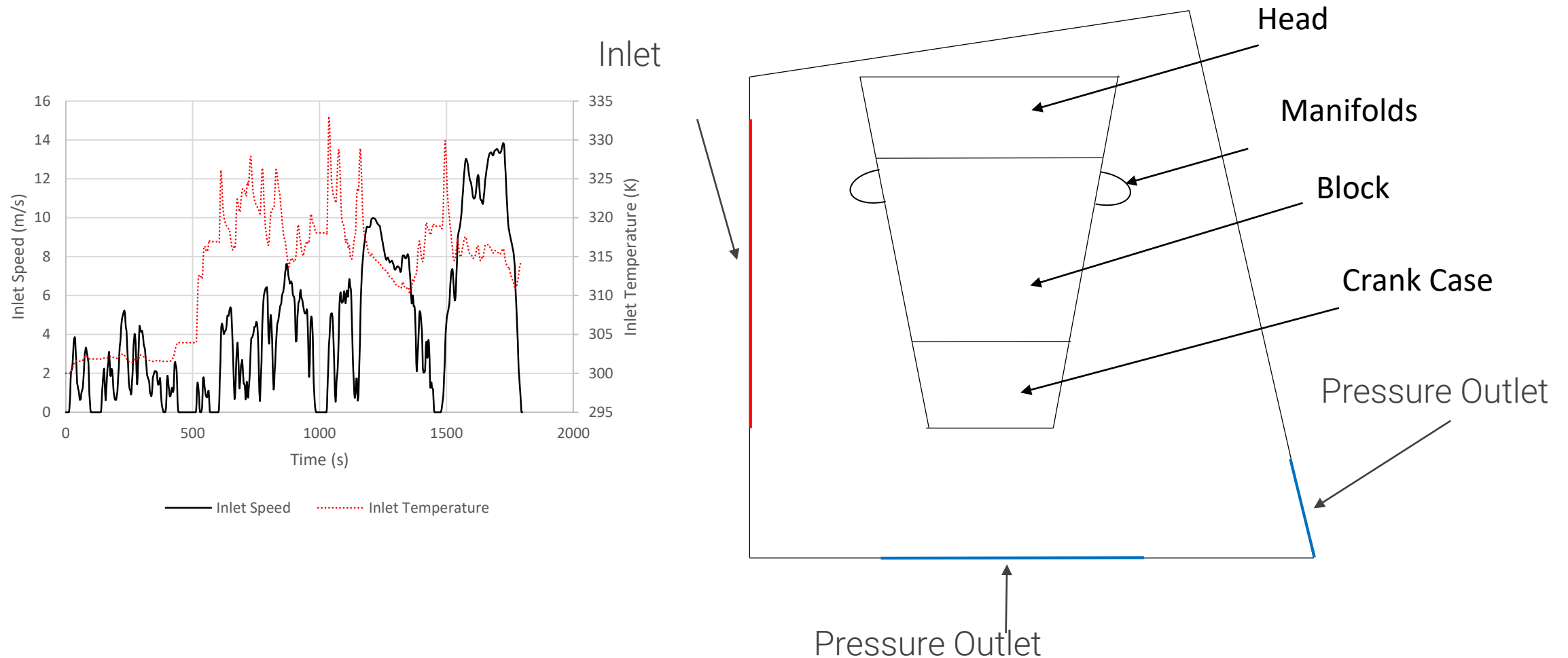


Case Study Introduction



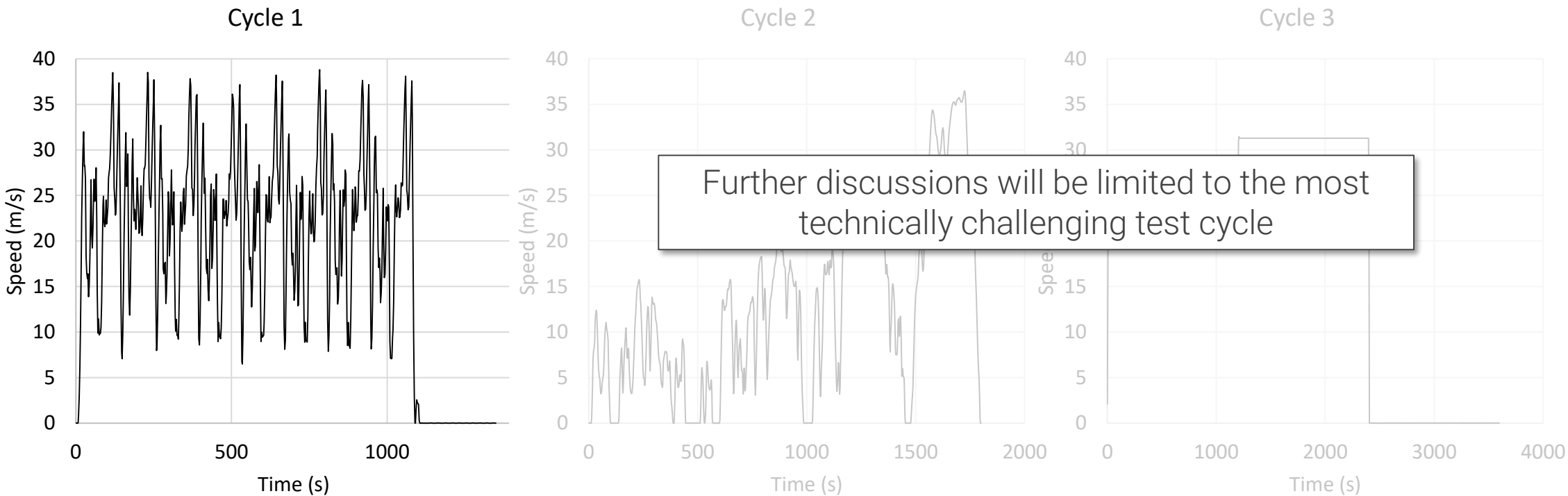
A Comparison of Strategies for Simulating Vehicle Heat Protection Test Cycles in 3D, SAE TMSS, 2018

Case Study Introduction



A Comparison of Strategies for Simulating Vehicle Heat Protection Test Cycles in 3D, SAE TMSS, 2018

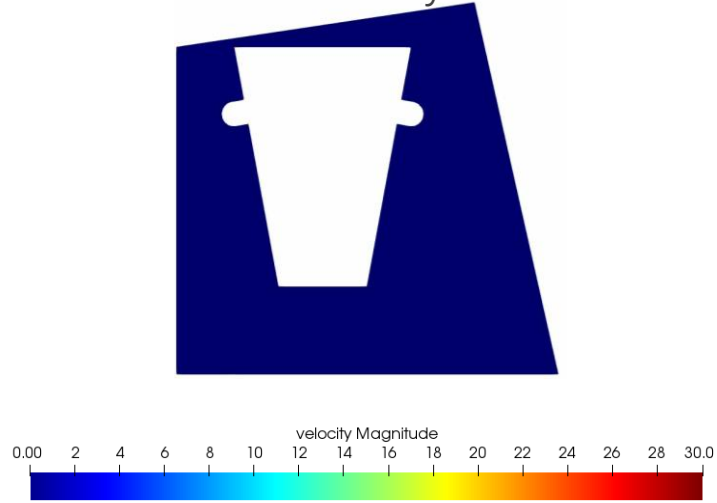
Case Study Introduction



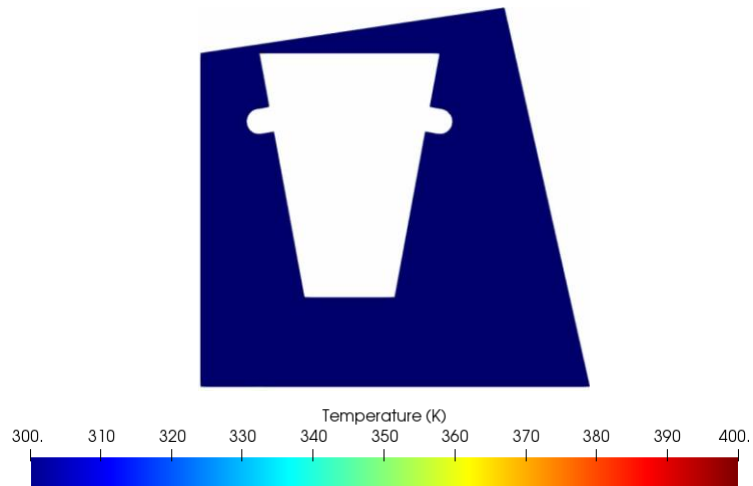
Duration (s)	1350	1800	3600
Max Speed (m/s)	38.8	36.5	31.4
Avg. Speed (m/s)	18.0	12.9	18.6
Volatility	2.98	0.27	0.06

CHT Simulation

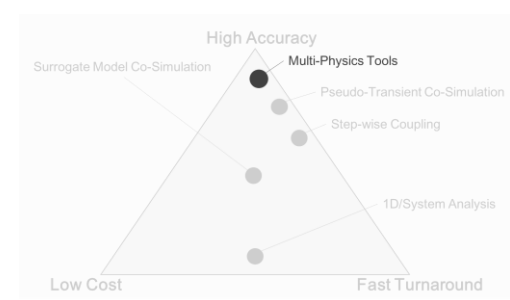
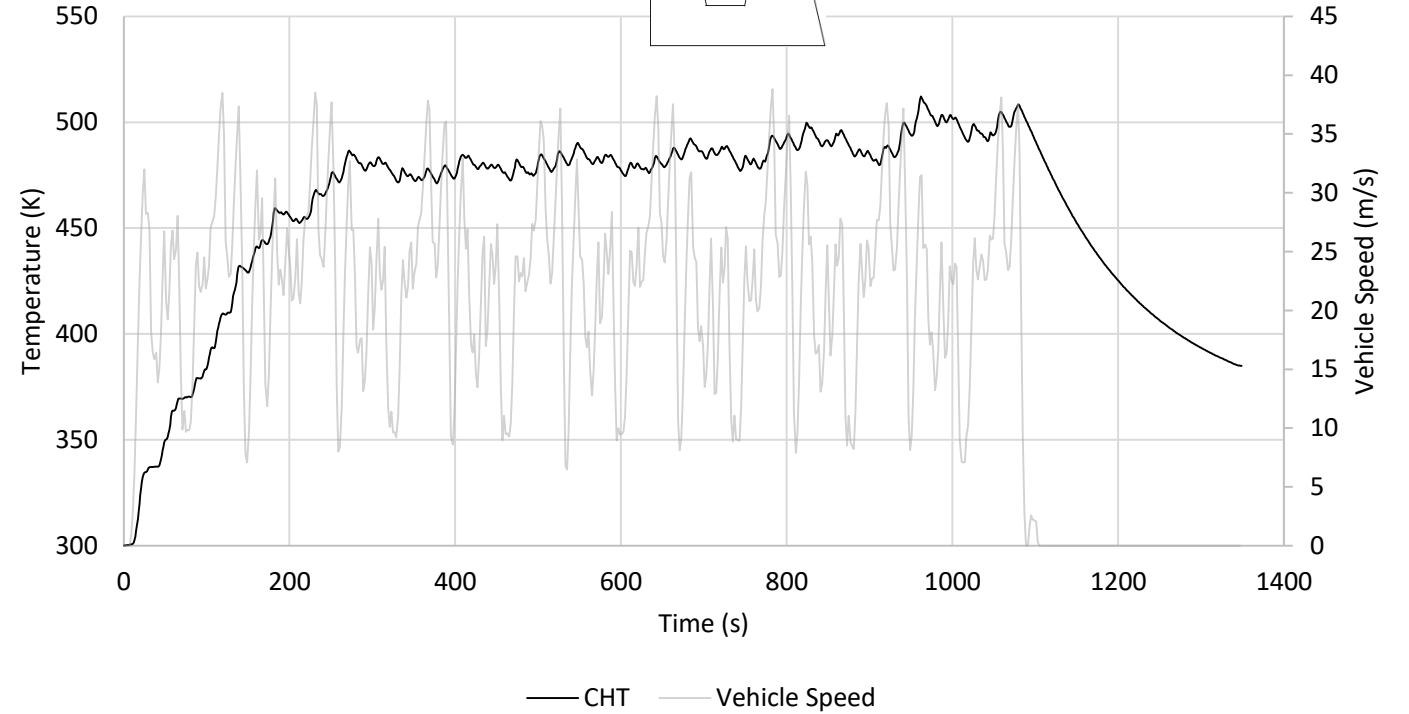
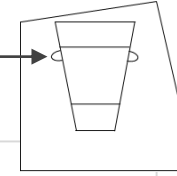
Velocity



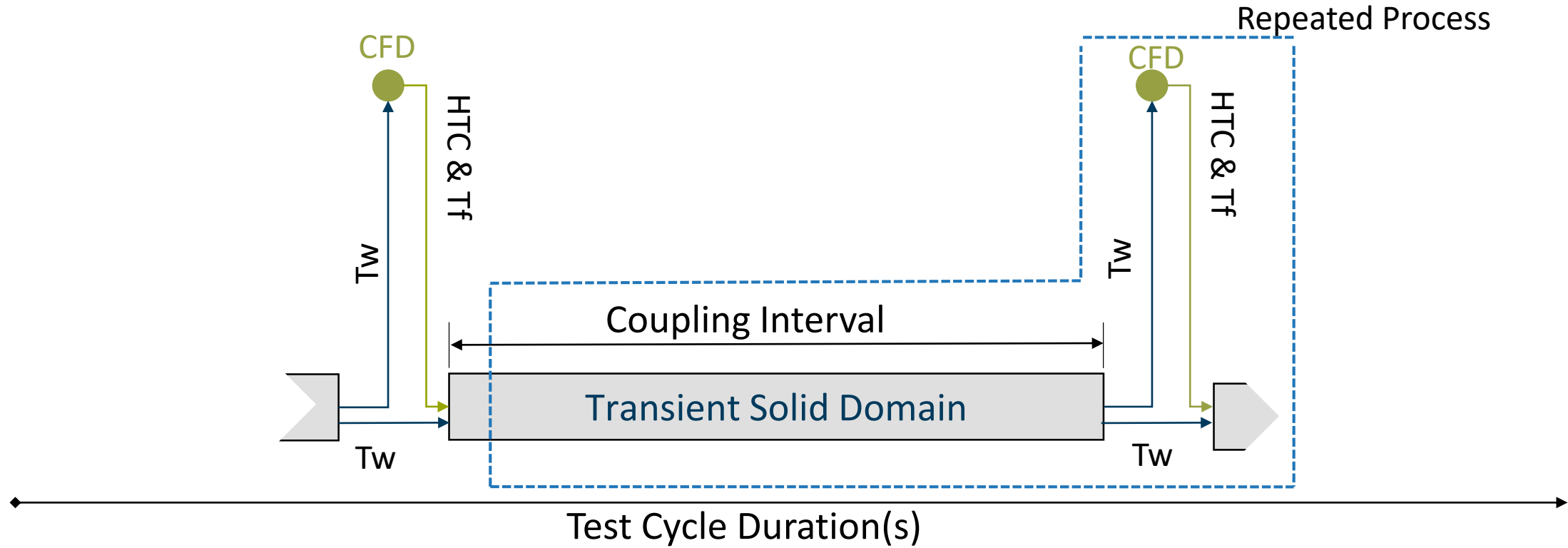
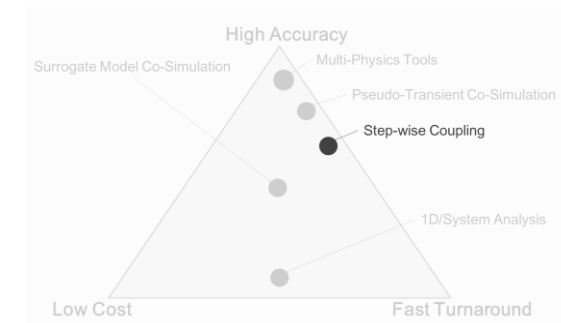
Temperature



6 node average

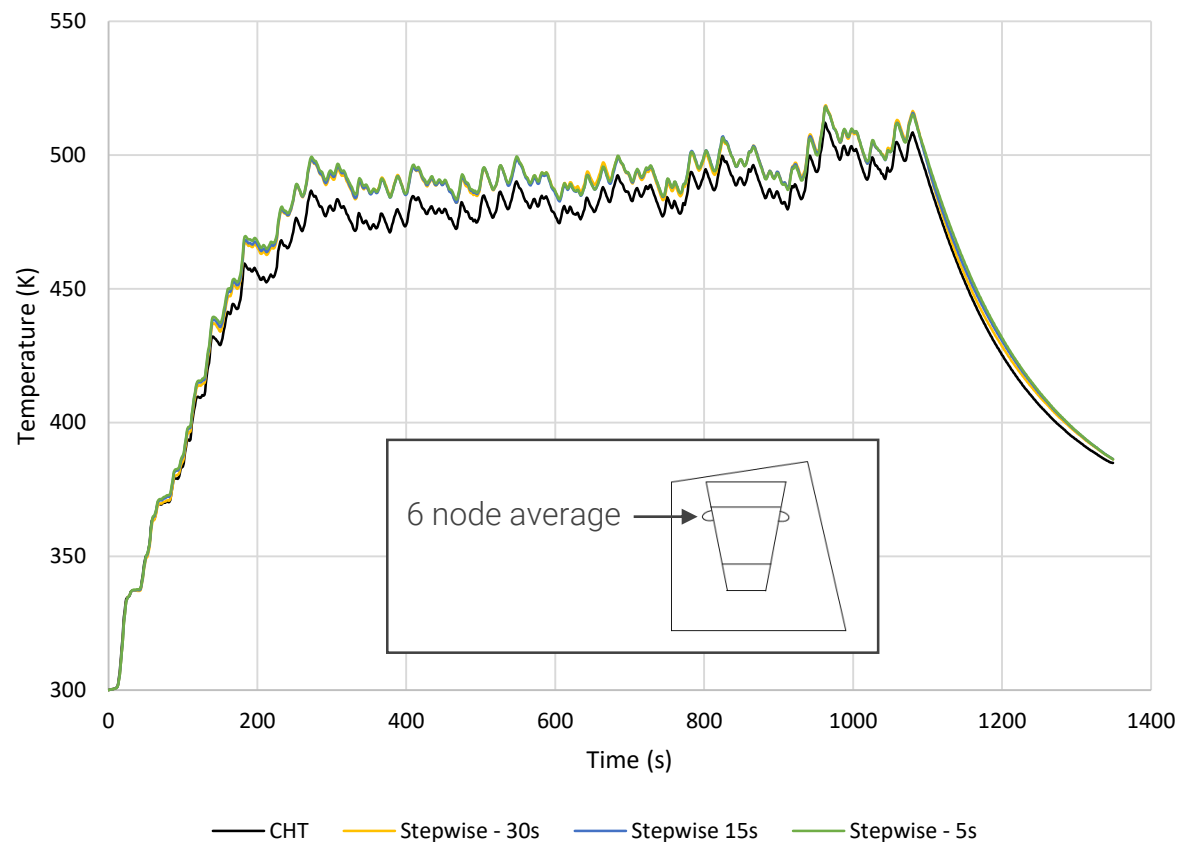


Step-Wise Coupling

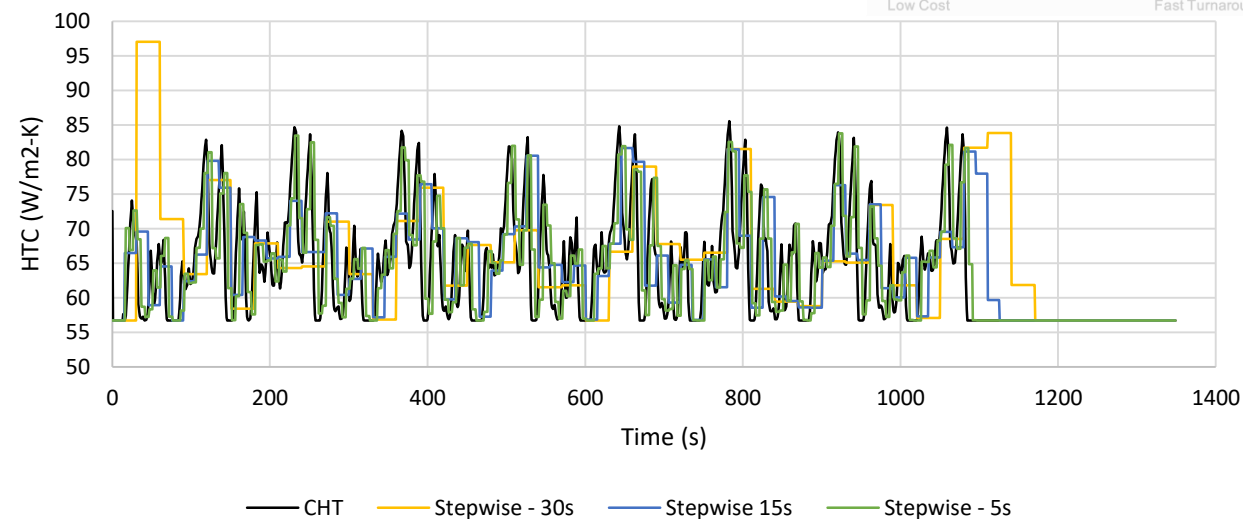


Step-Wise Coupling

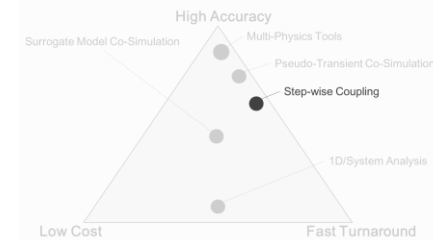
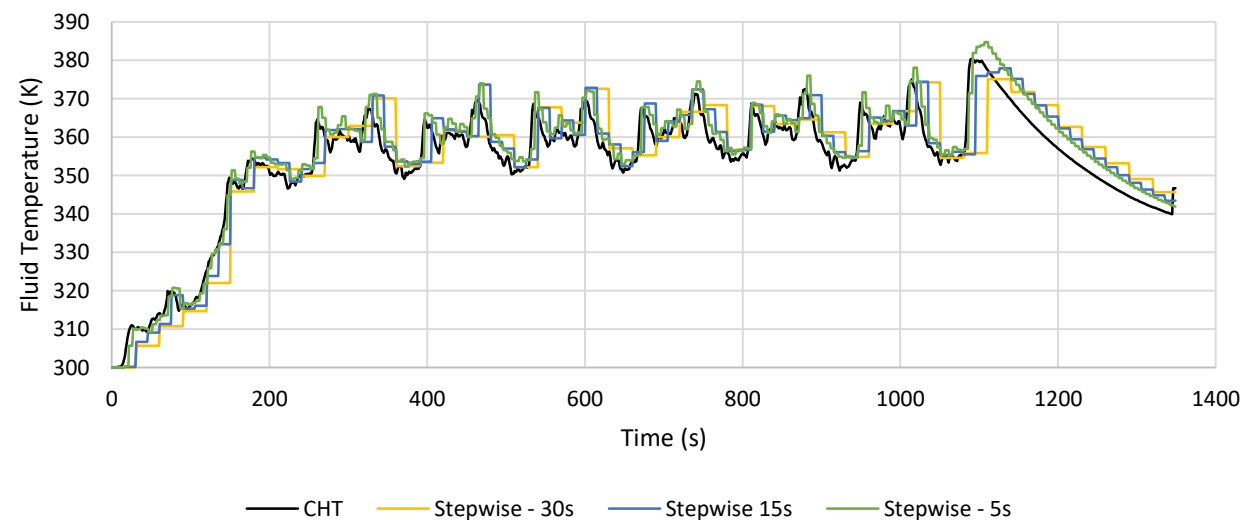
Wall Temperature



Heat Transfer Coefficient

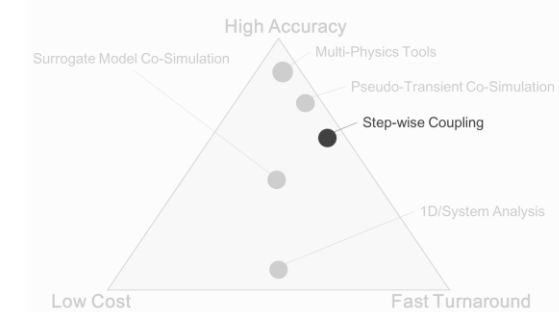
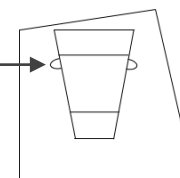


Fluid Temperature

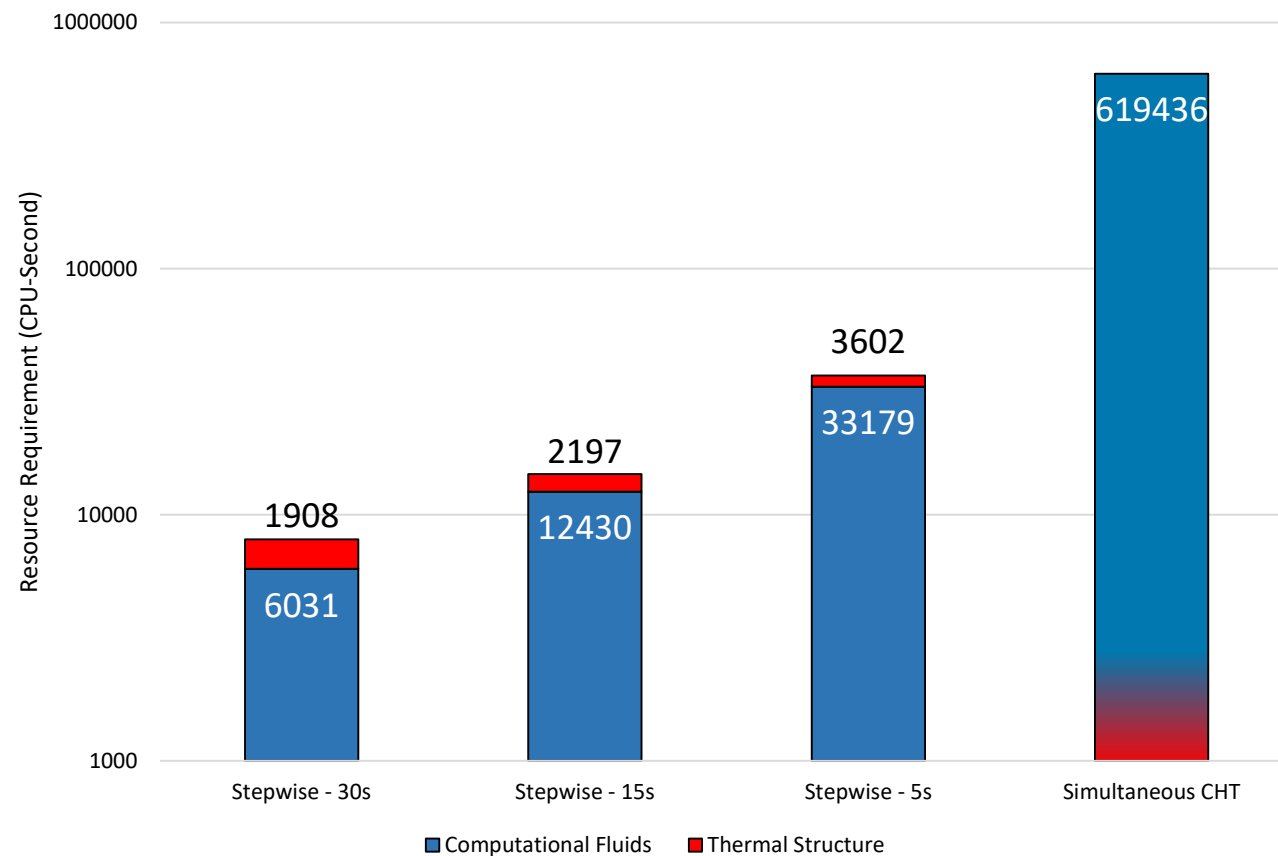


Step-Wise Coupling

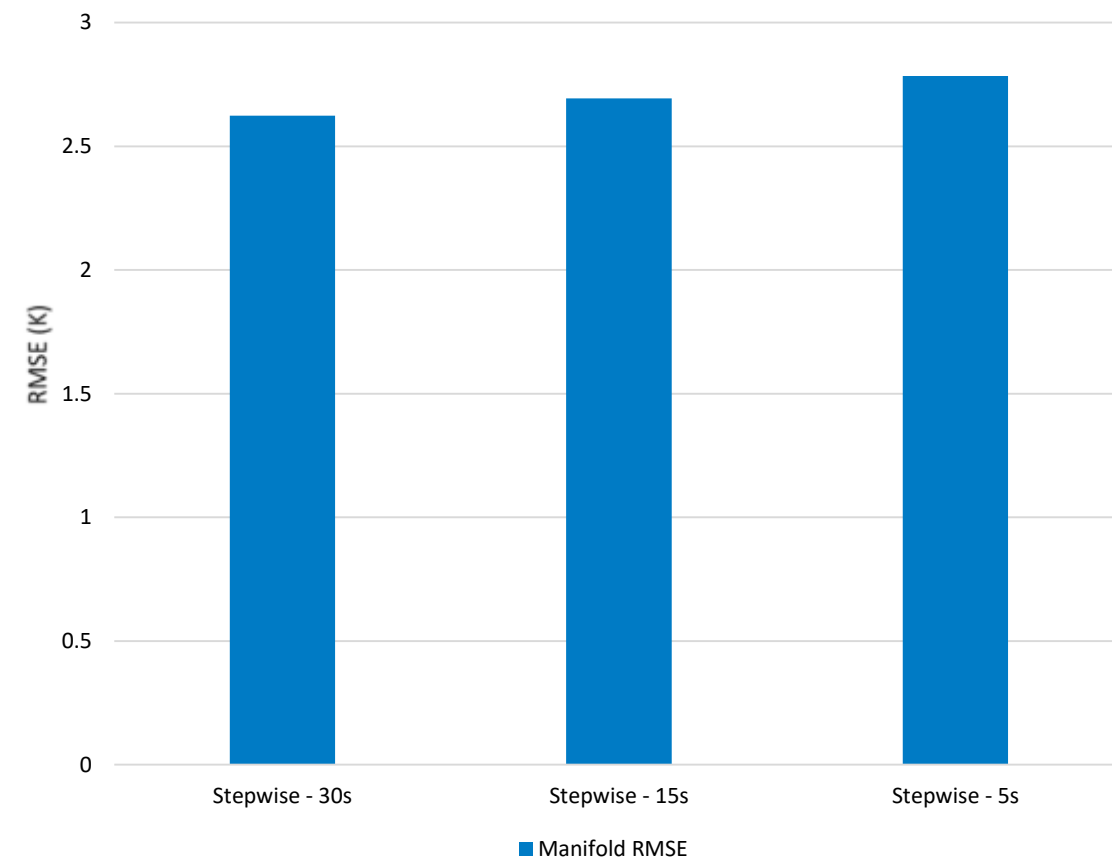
6 node average



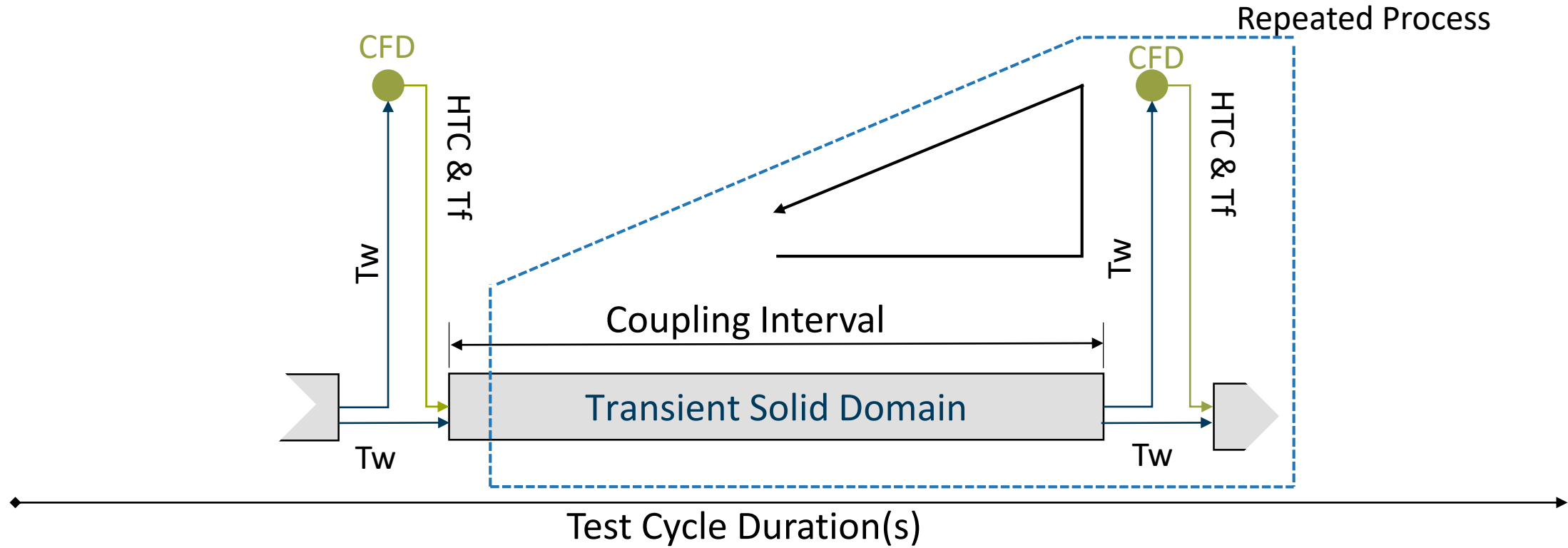
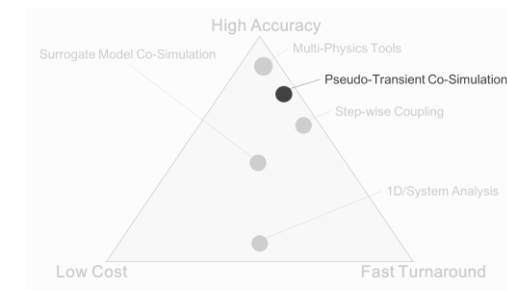
Simulation Times



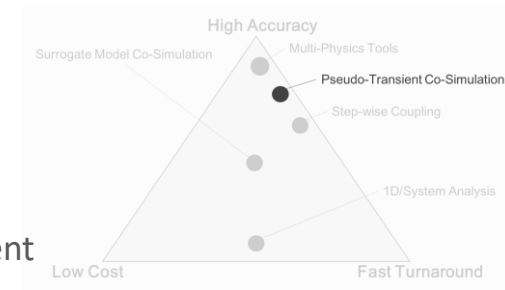
Manifold RMSE



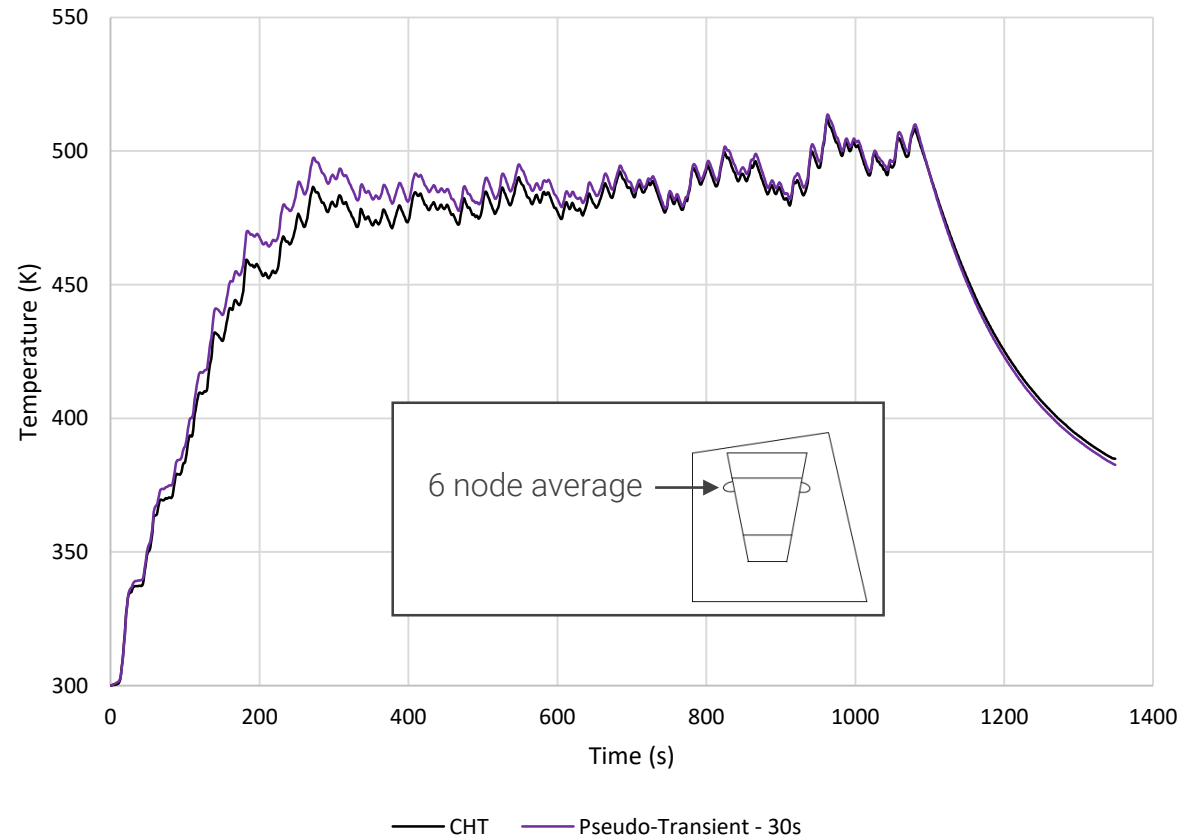
Pseudo-Transient Coupling



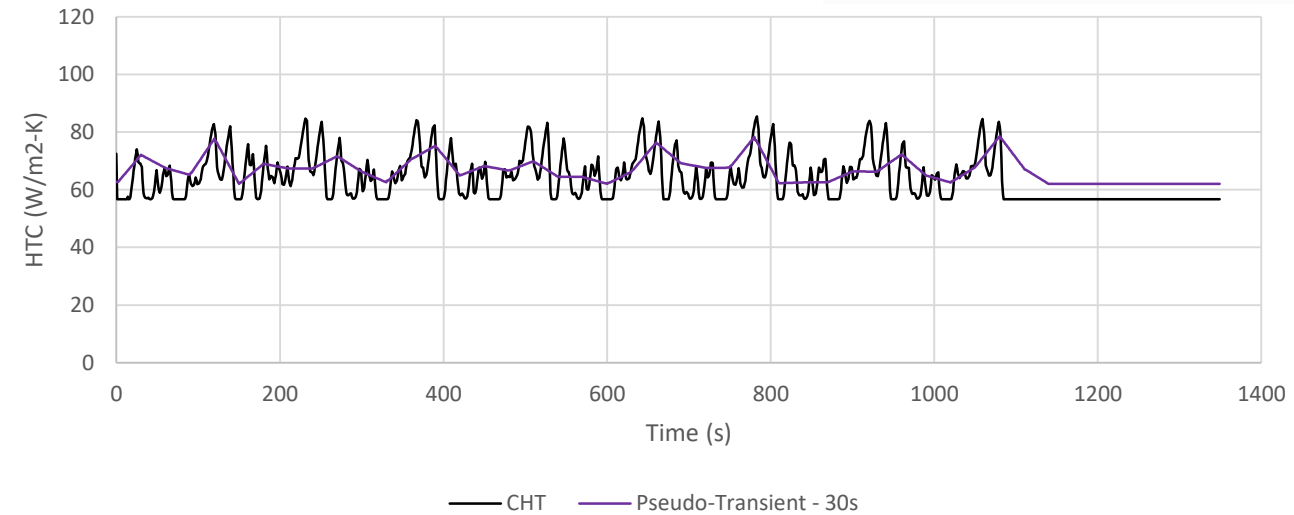
Pseudo-Transient Coupling



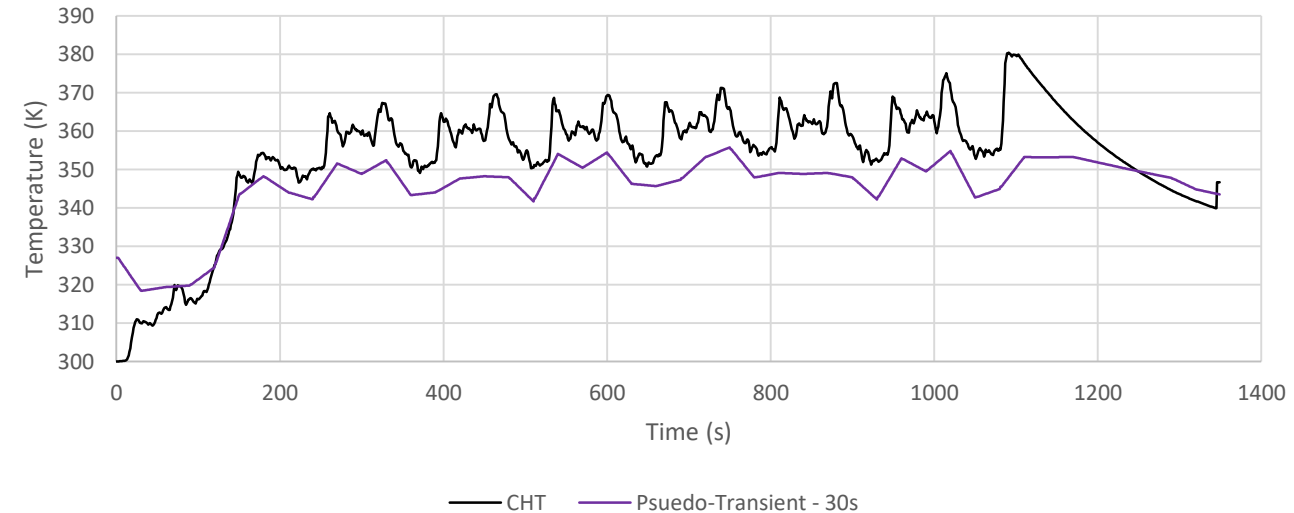
Wall Temperature



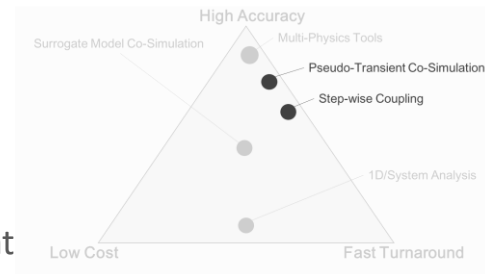
Heat Transfer Coefficient



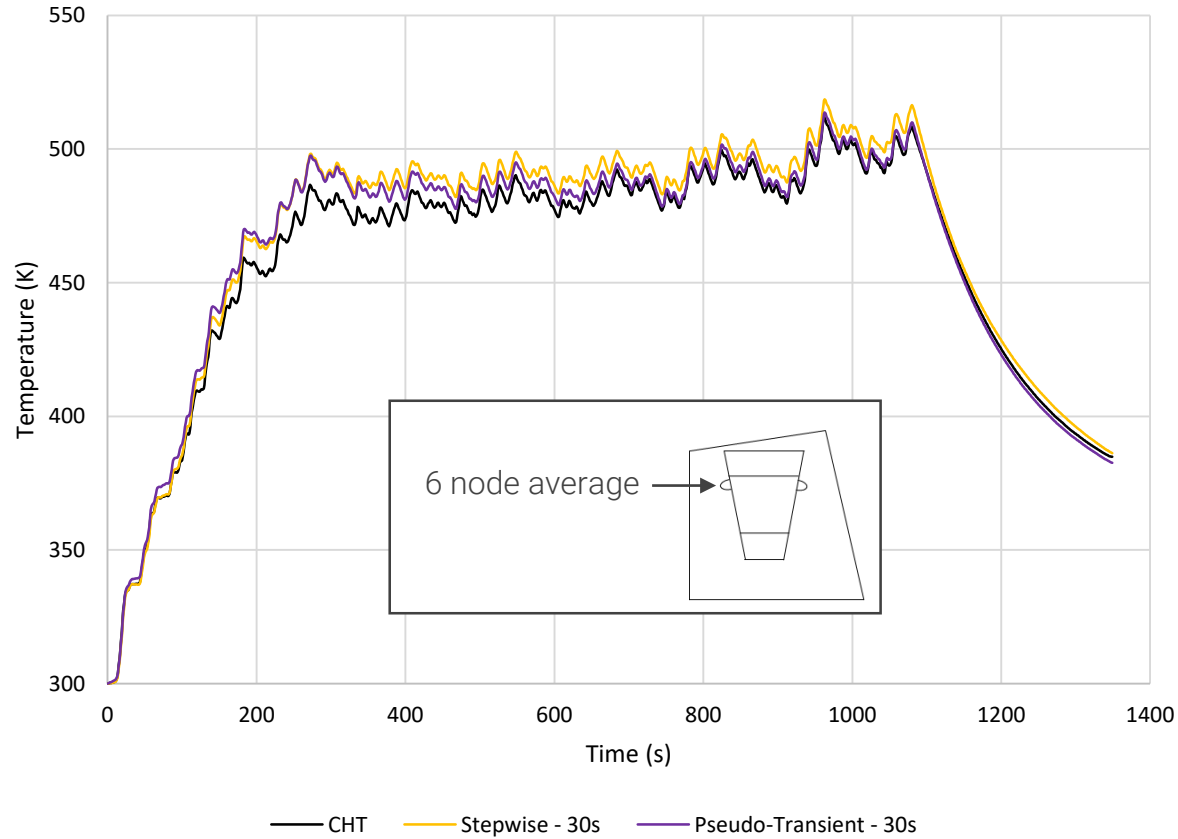
Fluid Temperature



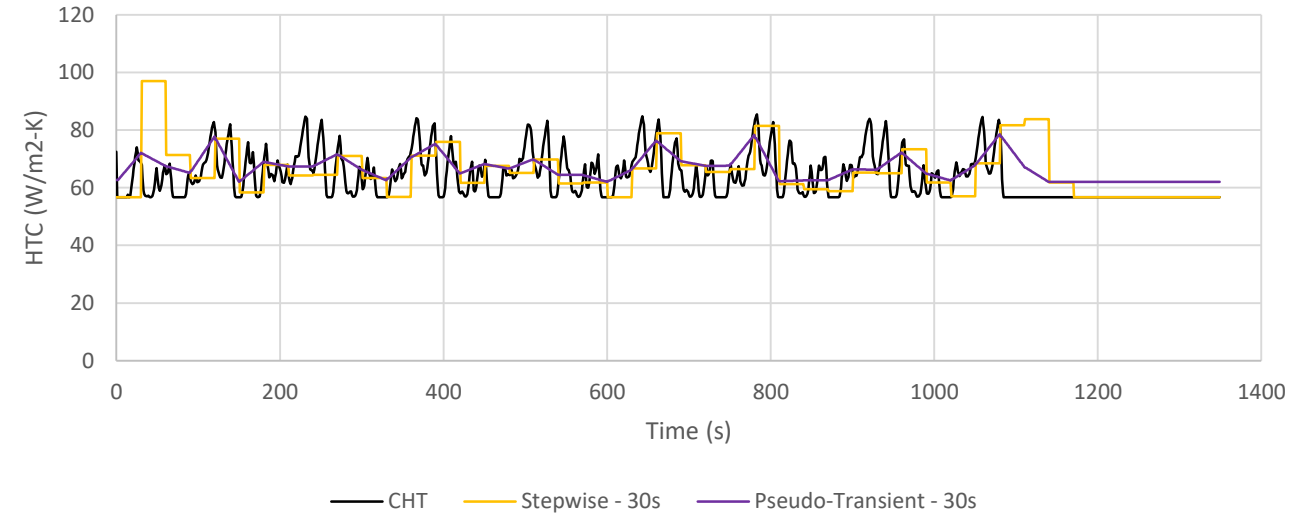
Pseudo-Transient Coupling



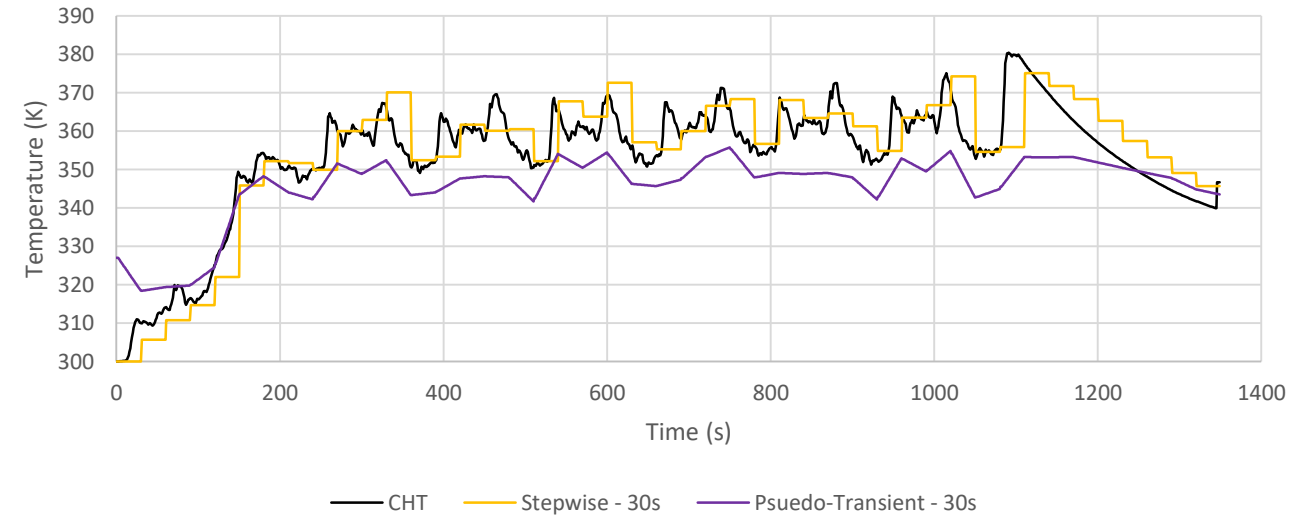
Wall Temperature



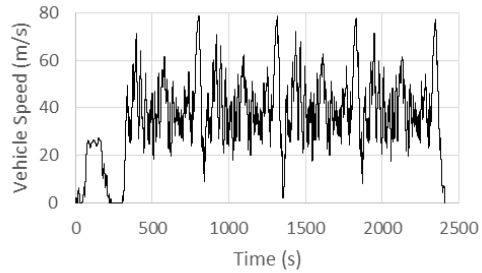
Heat Transfer Coefficient



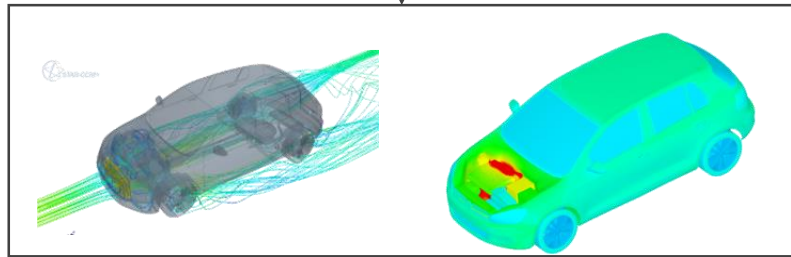
Fluid Temperature



Surrogate Model Co-Simulation



Select a set of representative vehicle parameters



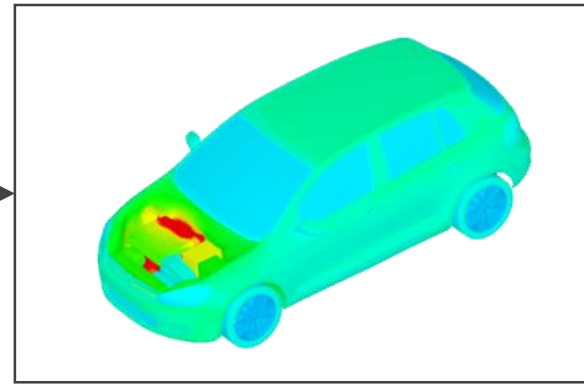
Calculate the steady state CHT solution at each selected speed

Extract HTC & T_f

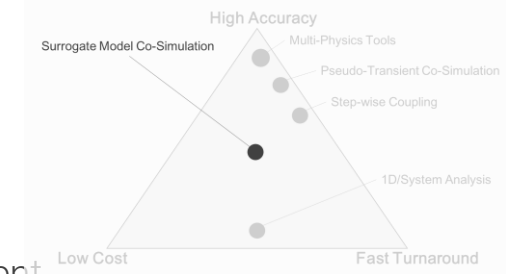
Surrogate Model

Make the samples continuous over the range of speeds with a surrogate model

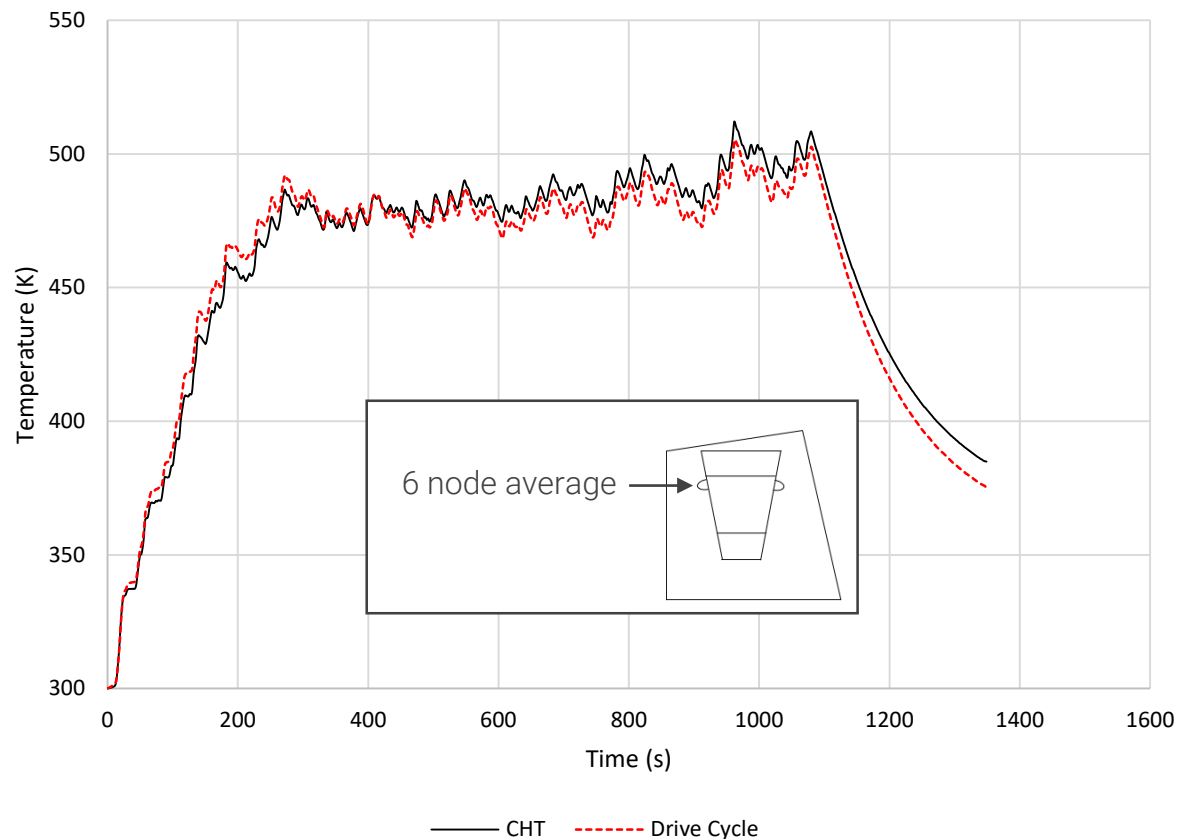
Solve transient thermal simulation



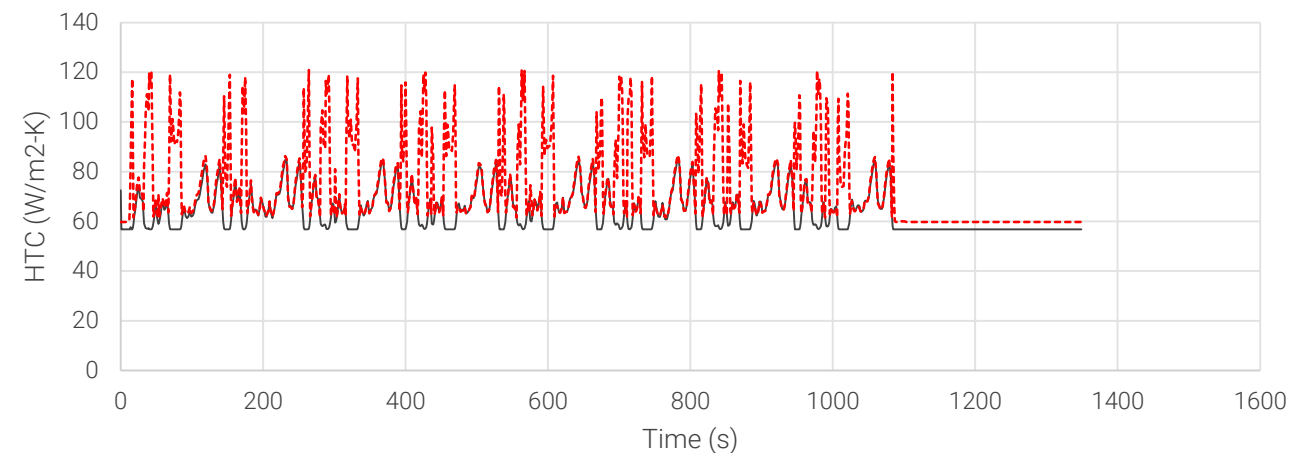
Surrogate Model Co-Simulation



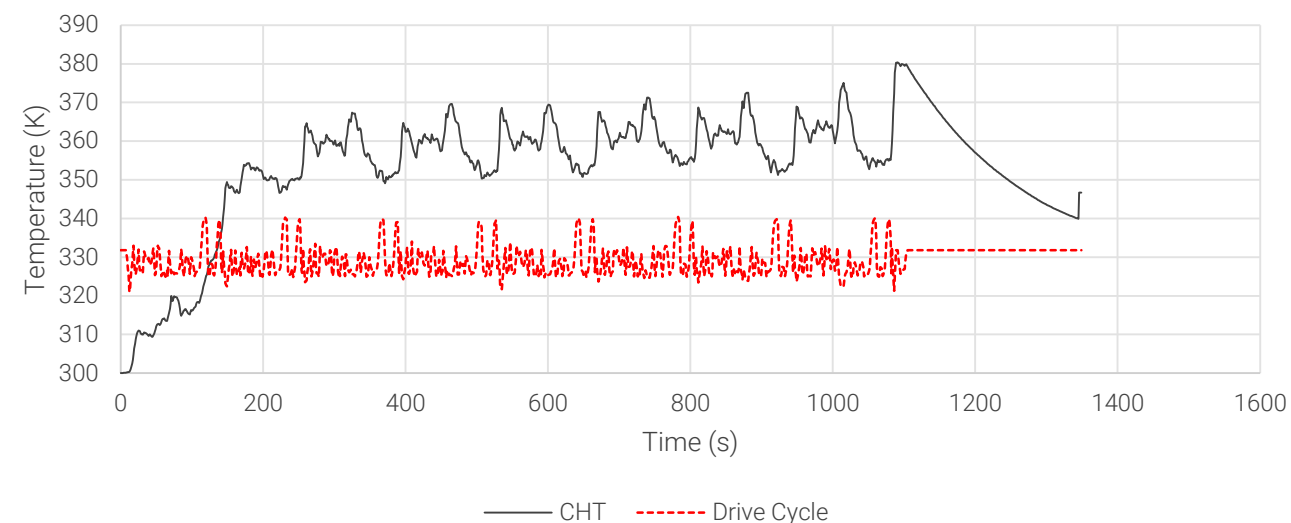
Wall Temperature



Heat Transfer Coefficient

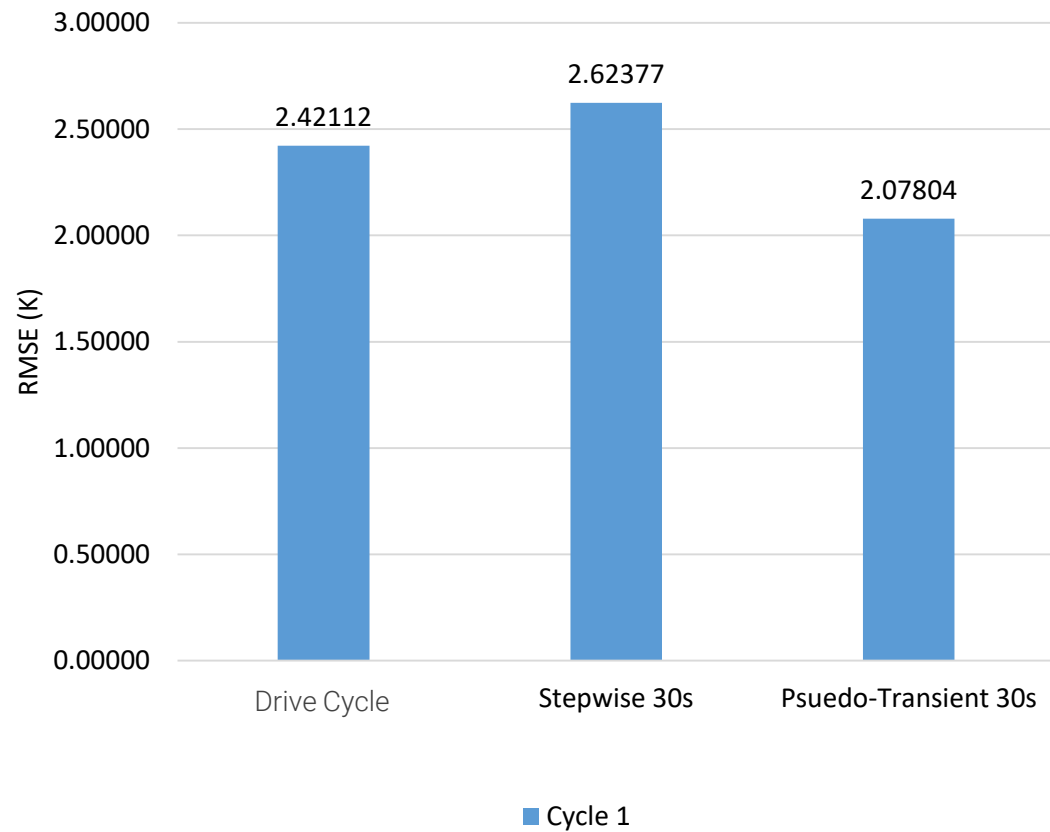


Fluid Temperature



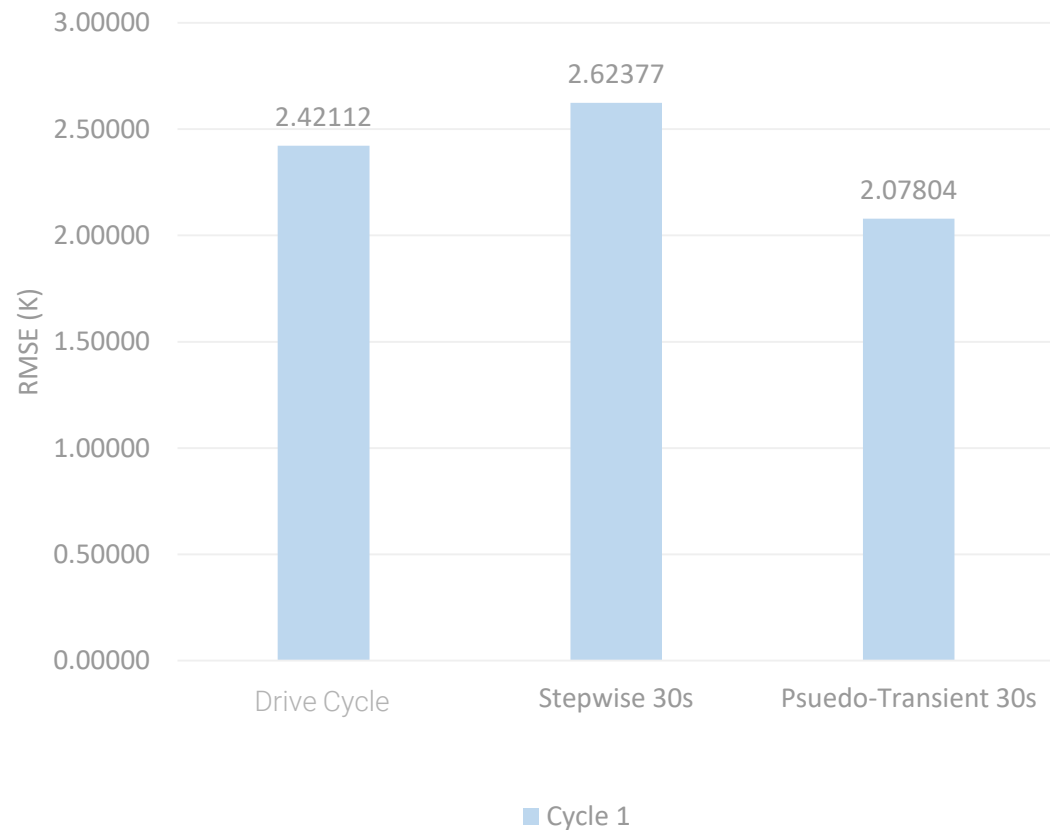
Case Study Conclusions

Manifold Prediction Accuracy

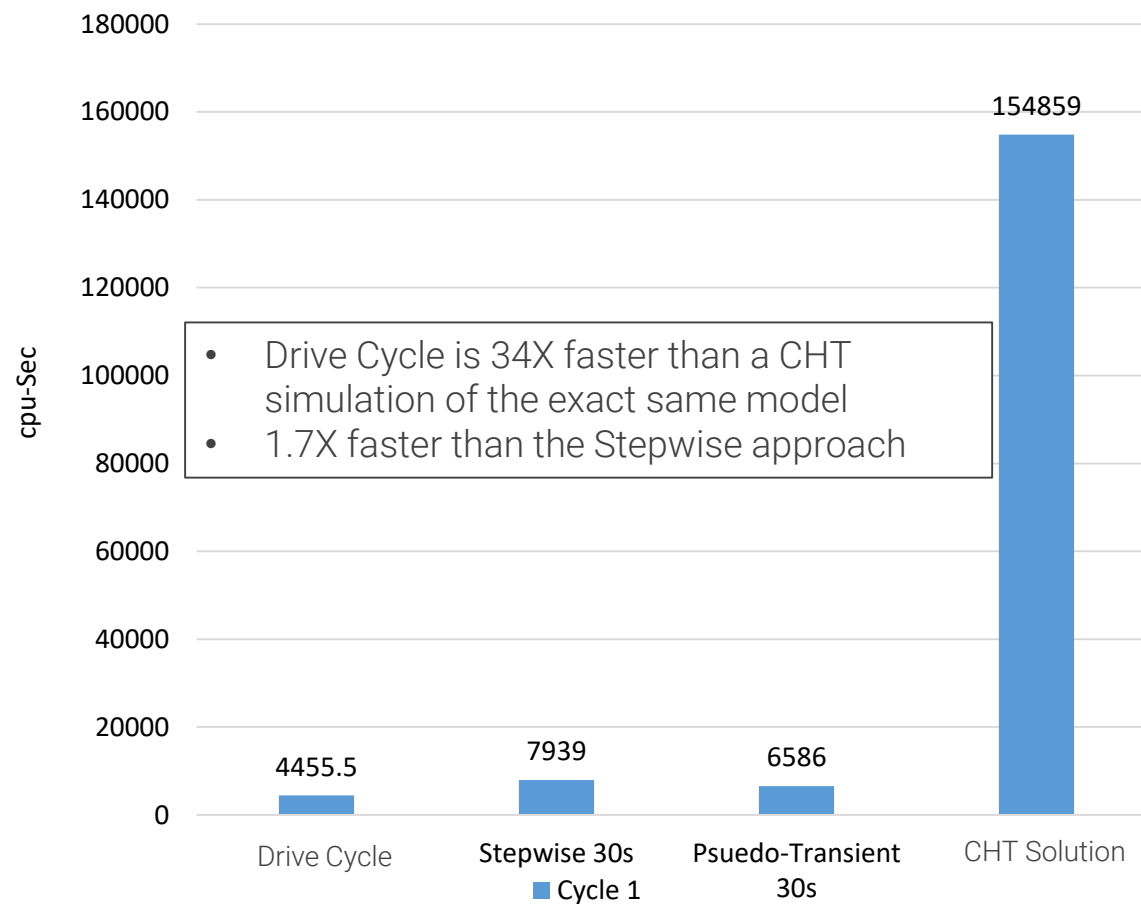


Case Study Conclusions

Manifold Prediction Accuracy



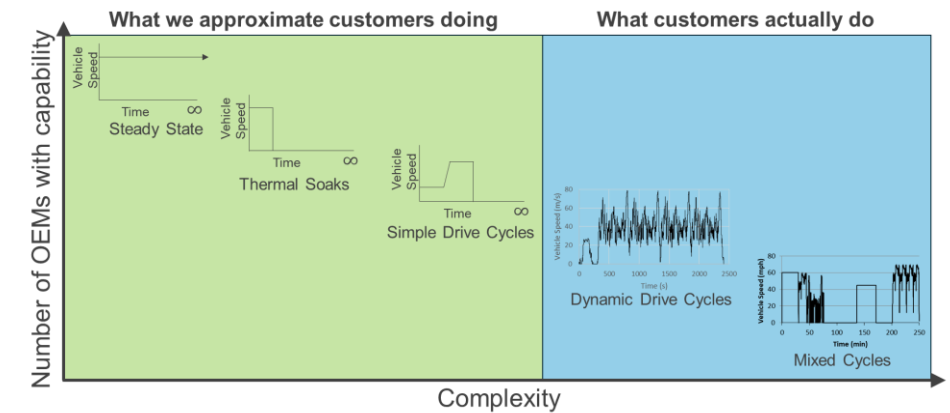
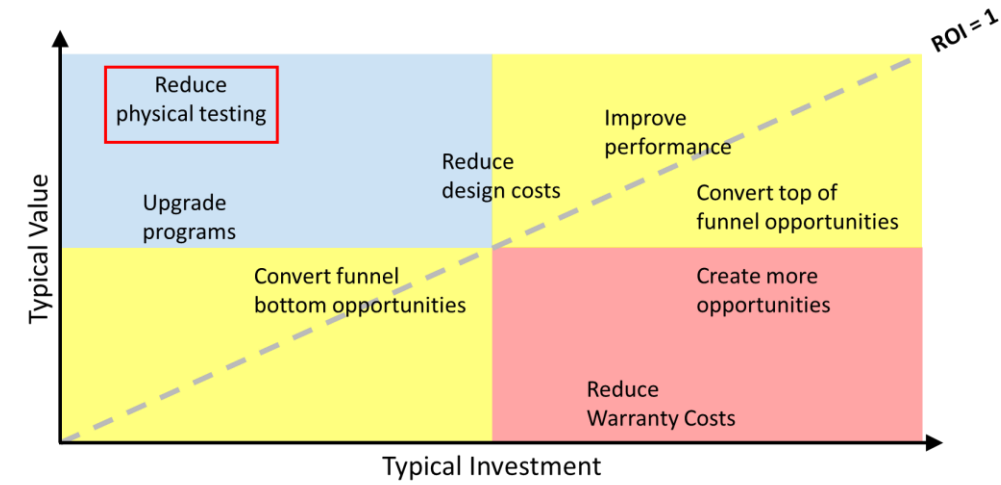
Cycle Compute Cost



Conclusions

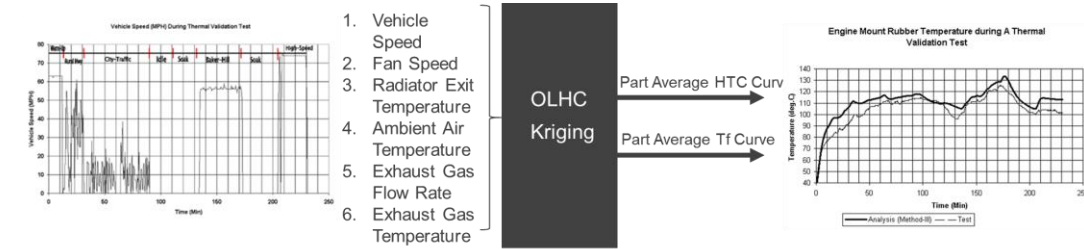
Conclusions

- Reducing physical testing costs create growth opportunities
- Transient modeling capabilities are huge opportunity to reduce testing costs
- TAI's Drive Cycle Extension enables the simulation of any test cycle

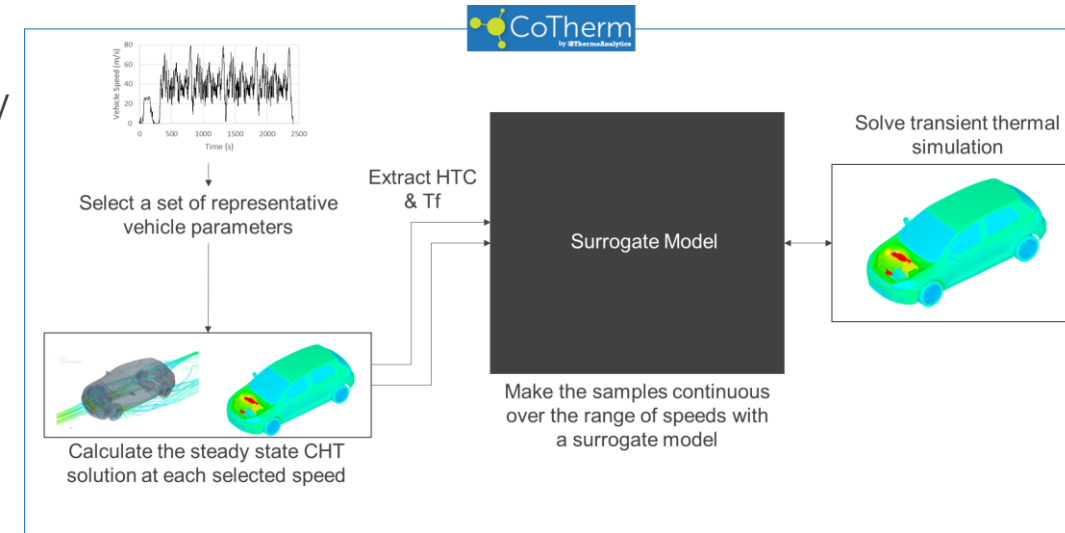


Conclusions

- Reducing physical testing costs create growth opportunities
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 - TAL's Drive Cycle Extension enables the simulation of any test cycle
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- There is a long history of surrogate modeling being used in engineering design
 - Proven to be accurate and low cost
 - TAL's CoTherm software makes the process simple to use, easy to deploy and highly automated

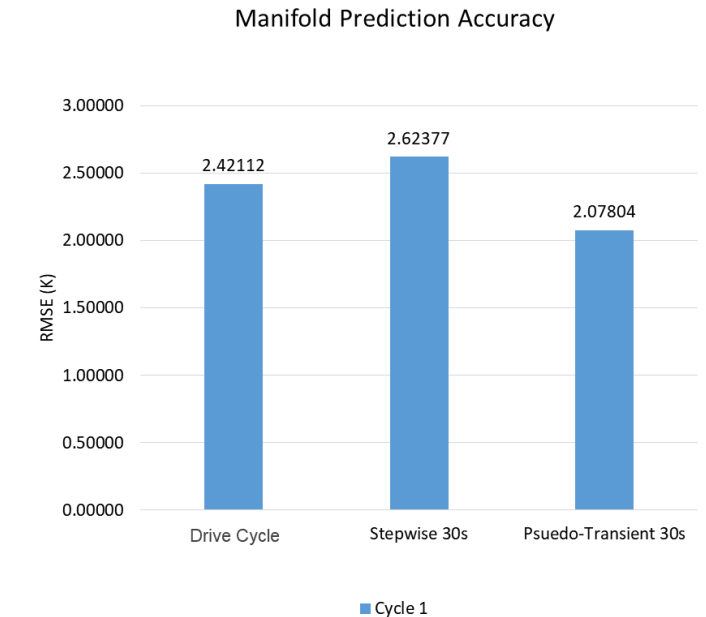
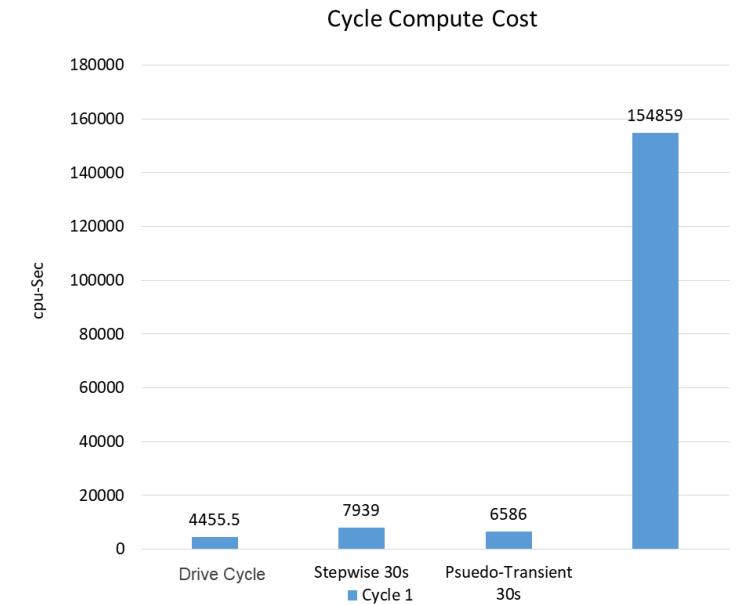


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Conclusions

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- TAL's CoTherm software makes the process simple to use, easy to deploy and highly automated
- The use of surrogate models for convective boundary conditions greatly reduces simulation times
- Surrogate models are as accurate as other coupled simulation methods
- The reduction in run time allows greater number of design studies





Thank you

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